

# Innovative Progress in Solar Chimney Power Plant Efficiency Improvements: A Comprehensive Review

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## Abstract

Utilizing solar chimney power plants (SCPPs) for manufacturing clean and environment-friendly energy has drawn a lot of attention and has (over the passing decades) become one of the most promising solutions in the solar energy field. Low efficiency, construction difficulties and other required improvements have encouraged researchers to work on this system. Many researchers put their efforts into proposing an optimized configuration for the main components, whereas others have proposed innovative ideas and add-on accessories to improve solar chimney power plants from an efficiency or construction viewpoint. This paper provides a comprehensive review of the past few decades, and includes theoretical, experimental and numerical studies focused on optimizing the main characters of the system such as the chimney, collector and power conversion unit (PCU) together with other recently suggested innovative ideas and alternative technologies to improve solar chimney power plants efficiency. Concurrently, other researchers focused on hybrid solar chimney power plants to produce desired by-product such as distilled water and so make SCPPs more practical.

## Highlights

- Different types of solar chimney power plant systems are reviewed in this paper
- Various techniques toward system improvement have been categorized and discussed
- Developments in hybrid solar chimney power plant systems are reviewed
- Experimental, numerical and theoretical studies are summarized and main effective results are pinpointed
- Key important innovative ideas and strategies for improving basic components are studied alongside integrated apparatus inserting layouts

**Keywords:** Solar Chimney Power Plant; Renewable Energy; Solar Collector; Hybrid Solar Chimney; Accessories and Alternative Technologies.

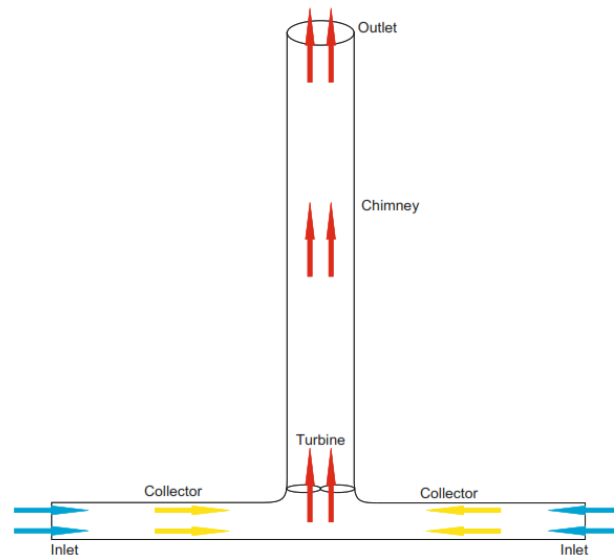
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45 **1. Introduction**

46 At this time, what with the noticeable growth in energy consumption all over the  
47 world, the limitations of energy resources, the environmental problems of fossil  
48 fuels energy and the hazards of climate change, choosing a clean, reachable and  
49 abundant energy resource are becoming a vital necessity [1, 2]. Population growth  
50 and increasing living standards are causing an ever faster-growing energy demand.  
51 Added to which, fossil fuel depletion and Green House Gases (GHG) pollution are  
52 becoming, more than ever, a burden on the environment. Furthermore, this shortage  
53 of current energy resources and the global warming concerns has forced  
54 governments and decision-makers to effect a change to renewable and sustainable  
55 energy resources. Hence, in the last few decades, designing a sustainable and  
56 efficient system to produce power has, more than ever before, become an essential  
57 research issue because access to a free and durable source of energy is necessary for  
58 progress[3]. Considering all energy resources – fossil fuels, nuclear energy,  
59 geothermal, hydro, biomass, solar energy and other types of resources, almost all of  
60 them have some detrimental effects on the environment, but solar energy is more  
61 available, durable, has limitless energy potential, and more importantly, has causes  
62 minimal damage to the environment. In this respect, solar chimney power plant  
63 systems (SCPPs) use solar radiation for power generation and consist of three basic  
64 components: a collector – generally a huge circular and transparent roof [4], a  
65 chimney or tower –a super-tall tube, and a power conversion unit containing a  
66 turbine to convert kinetic energy into electricity. The chimney is installed in the  
67 center of the air collector and utilises the buoyancy effect so warm air rising through  
68 the chimney once the air temperature within the ‘greenhouse effect’ collector has  
69 risen sufficiently [5] operates the turbine, as shown in Fig. 1.



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Fig. 1. Solar chimney power plant (SCPP) schematic

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Multiple energy conversions take place in this form of power plant, converting from the thermal energy of the sun into kinetic energy of flowing air and finally into electricity. Despite the simplicity of the principle, several issues exist, the foremost

75 being their low efficiency [5]. In addition, there is the problem of the intermittency  
76 (inherent of such energy sources) on rainy or cloudy days [6]. Notwithstanding,  
77 many obstacles have been overcome at some level with the application of new  
78 concepts, nonetheless, there is still room for improvement – which has kept this  
79 topic as a center for research attention. With lower construction and maintenance  
80 costs, simpler technologies, cheap materials and little (if no) need for a high  
81 specialist input (particularly full time), this energy system have become an  
82 interesting technology for many countries [5] – particularly in remote areas and  
83 desert climates with their high solar irradiances and no translation losses.

84 There have also been numerous published works on SCPPs due to its capability for  
85 industrial and urban applications including reviews, and numerical and  
86 experimental studies [1, 7-21]. Nonetheless, this present review introduces a more  
87 detailed, updated and comprehensive information approach related to the modern  
88 developments in this technology. An applicable and innovative study including both  
89 experimental and analytical studies is presented in this work to cover all the recent  
90 studies performed on enhancing the efficiency of SCPP, necessary because  
91 innovation plays a pivotal role in their progress.

92 Moreover, because of their low efficiency, researchers have compensated for this  
93 by coupling the SCPPs with other units, which then results in hybrid plant capable  
94 of the desalination of water; generation of power; drying products; heating and  
95 ventilation and etc. Hence, because in recent years the integration of SCPPs with  
96 other units has become a focus of attention, this review also considers hybrid SCPPs  
97 to identify other gaps for future studies.

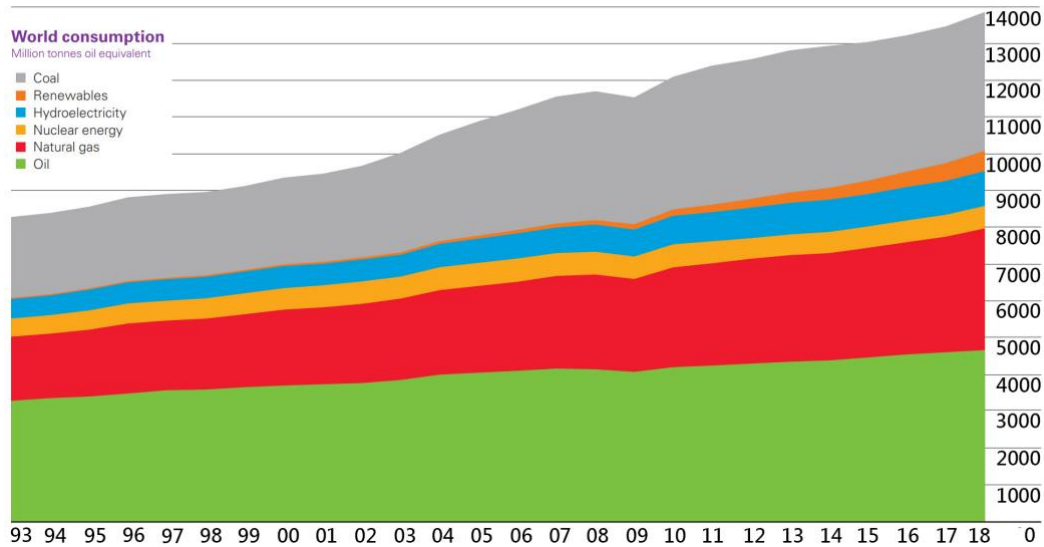
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### 99 1.1 Renewable Energy and the Environment

100 The high rate of population growth coupled with a remarkable growth in  
101 developments and lifestyle standards, has resulted in a rapidly increasing  
102 global energy and water demand during the last couple of decades, as shown  
103 in the Fig. 2 [22]. Coupled with this is the problem that, currently, fossil fuels  
104 are the world's dominant fuel source and it is this proportion of usage  
105 (compared to other resources), which leads to excessive GHG emissions.  
106 These, in turn, exert several profound, but negative influences on the  
107 environment which promote regrettable Worldwide changes, such as receding  
108 glaciers, earlier plant blooming, ocean acidifications, killer heat waves, even  
109 butterflies retreating up mountainsides [23]. Notwithstanding, all nations will  
110 encounter precipitation shifts that may vary from region to region  
111 [23]. Therefore, in order to help control these impacts, will require utilizing  
112 alternative choices for our energy resources, such as renewable energy  
113 resources, which noticeably reduce some of the detrimental effects of GHG  
114 emissions [23, 24].

115 Fig. 2 represents the world energy consumption in 2019 in million tones oil  
116 equivalent (MTOE). World energy consumption increased by about 3% in  
117 2018. Gas had the largest increase, followed by renewable energy [22]. From

118 the same figure also shows that oil remains the world's leading fuel, making  
 119 up a third of the total consumption. In 2009, all of the fuels, except oil, fell  
 120 moderately. Renewable energy consumption experienced a gradual increase.



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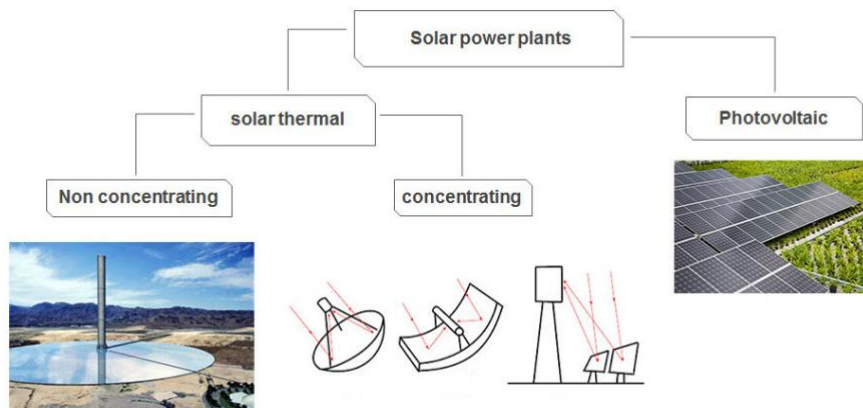
Fig. 2. World consumption in Million tonnes oil equivalent[22]

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## 124 1.2 Solar Energy and Solar Chimney

125 It has been claimed that solar power could be used to reverse the damage being  
 126 done by other environmentally-damaging energy production methods [24]. It  
 127 is one of the most promising renewable energy resources, it is both clean and  
 128 safe [24], and amongst all of solar power technologies (depicted in Fig. 3), PV  
 129 cells play a great role within the market to supply power demands. However,  
 130 they are still exhibit low efficiency, high-temperature drop-off, and are  
 131 vulnerability to harsh winds. All of which remain a problematic issue. In  
 132 addition, concentrating solar power plants are strictly dependent on direct solar  
 133 irradiance, something that SCPPs can turn to their advantage, because they do  
 134 not depend on just radiation that coming from the direction of the sun. Another  
 135 further problem of solar energy power production devices is their  
 136 intermittency, but this difficulty could be handled to some extent in SCPP  
 137 systems by utilizing a capability to harvest thermal energy stored in the ground  
 138 and so produce a limited amount of power even at night. Something not seen  
 139 in the other types of solar power plant. To improve this ability, some  
 140 researchers have also suggested exploiting phase change materials (PCM) –  
 141 (like paraffin [25, 26] and Glauber's salt (Sodium Sulfate Decahydrate) [27] – as  
 142 a latent heat energy storage medium to store more energy for nighttime power  
 143 generation. Sedighi et al. [28] investigated the effect of PCM porosity on the  
 144 SCPP performance numerically. On the other hand, Fadaei et al. [29] used an  
 145 artificial neural network to investigate the performance of SCPP, and Rafea et  
 146 al. [27] ran an experimental setup to investigate the effect of PCM material in

147 SCPP performance enhancement. Even so, simpler and cheaper concepts, like  
148 filled water tubes [30], could still enhance sustained power generation utilizing  
149 the earth surface soil's ability to act as a power storage device for a system.



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Fig. 3. Types of the solar power plant based on the working mechanism

152 The SCPP is a large-scale power generation unit which absorbs solar radiation  
153 and converts it into the power through the installed equipment [24, 31-33]. The  
154 multiple energy conversions taking place in SCPP are: solar radiation to  
155 thermal energy, thermal energy to kinetic energy, kinetic energy to mechanical  
156 energy, and mechanical energy to power, respectively [1].

157 The first chimney power plant was proposed by Cabanyes [34] in order to heat  
158 a house and generate electricity through an installed wind propeller, although  
159 the basic idea was not a new innovation as many years ago, Leonardo da Vinci  
160 designed a barbecue which worked using the basic idea of the updraught within  
161 a chimney [35]. However, the first actual prototype was constructed by  
162 Schlaich et al. [5] in Manzanares, Spain in 1982 as shown in Fig. 4. Their  
163 objective was to determine the efficiency of the system [36]. The height of the  
164 constructed prototype is 194 meters and collector diameter, tower diameter and  
165 collector inlet heights are 244, 10 and 1.8 meters, respectively. The total weight  
166 of the prototype is approximately 125 tons. Construction of the Manzanares  
167 power plant was an inspiration to start innovative researchers in other countries  
168 to study the feasibility, in places such as China [12, 15, 37, 38], Iran [39, 40],  
169 Mediterranean region [41], Algeria [42, 43] and other countries.

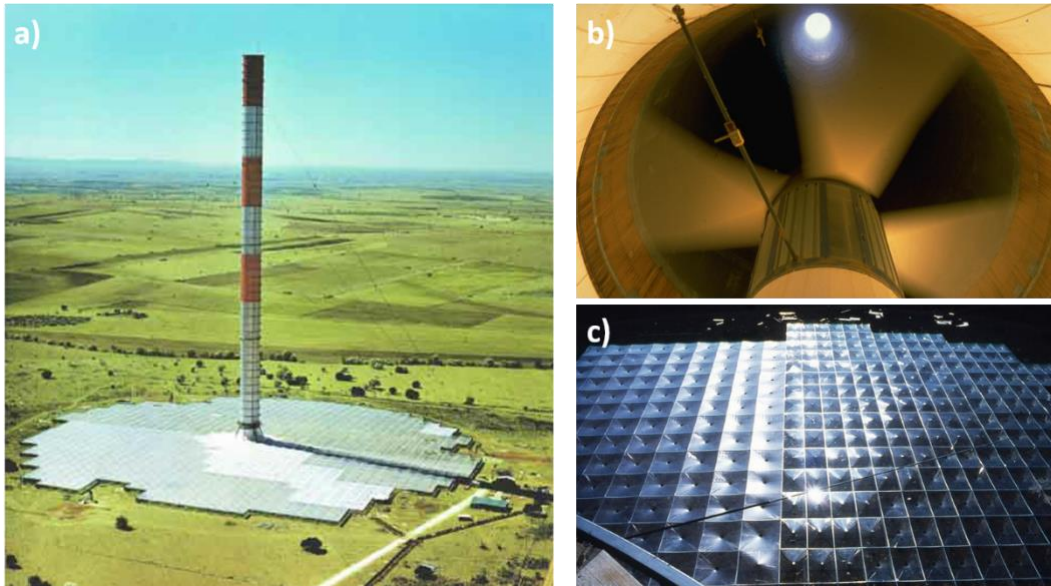


Fig. 4. Different sections of the solar chimney in Manzanares, Spain; a) side view of the SCPP, b) Turbine, c) Collector[44]

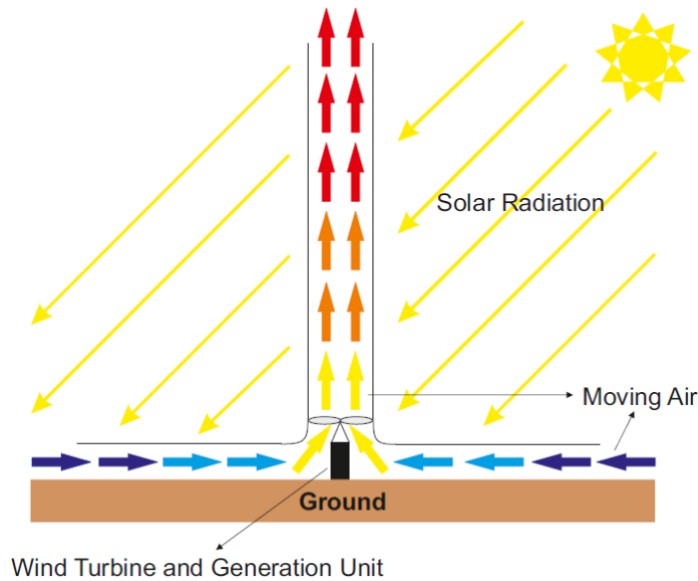
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## 2. Basic Components of the Solar Chimney Power Plants

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176 The SCPP (solar chimney power plant) is a system that converts both direct and  
177 non-direct irradiance into a clean reliable and environment-friendly power. This  
178 eco-friendly system is composed of a collector – that plays a role in absorbing  
179 irradiance and heating air by the greenhouse effect phenomenon, a large chimney –  
180 that plays an important part in conducting heated air through the turbine and  
181 atmosphere and a turbine as the final power conversion unit, as illustrated in Fig. 5.  
182 From these beginnings, there has been considerable focus on how to optimize the  
183 main components of SCPPs in order to harvest more efficient power.



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Fig. 5. Mechanism of solar chimney power generation

186 Many researchers have put a great effort into this area, exploiting experimental  
187 setups, theoretical models and numerical simulations to understand the  
188 mechanism and present an optimum amount of basic parameters such as:  
189 chimney height [45], chimney diameter [46], the divergence angle of the  
190 chimney [47], the ratio of height and diameter in the chimney[48], collector  
191 radius and collector inlet height [46, 48, 49], all of which have been considered  
192 as the most relevant parameters that influences a SCPP's performance. As a  
193 simple precept, Schlaich et al [50] stated that solar tower power output is  
194 proportional to the size of an imaginary cylinder that encircles the chimney  
195 inlet area and extends to the height of the chimney. However, although early  
196 studies reported that power output is directly proportional to collector area and  
197 chimney height, finding an optimum value for each parameter and the best  
198 configuration for a solar chimney remained as an argument for discussion, and  
199 there was not a complete investigation that included all the parameters.  
200 Additionally, limitations from a practical viewpoint of regional dependences  
201 and economic considerations should be considered before proposing an optimal  
202 efficient SCPP design.

### 203 2.1 Chimney

204 The part of the system that conducts heated air from the collector part to the  
205 atmosphere, utilizing the buoyancy effect causing by inside and outside of apparatus  
206 temperature differences has been called a 'chimney', 'updraft tower' 'or a solar  
207 tower'. Irrespective of the term, it is a gigantic tube that is sited at the center of the  
208 collector acting as the thermal engine for the plant. Despite the long tube, it has been  
209 claimed that the chimney has a low friction loss because of its suitable surface to  
210 volume ratios and so likened to a hydropower station pressure tube or penstock [50].  
211 Chimney efficiency is given by the following equation and depends on height in a  
212 particular case [24].

$$213 \quad \eta_{chimney} = \frac{g \cdot H}{c_p \cdot T_0}$$

214 Using the Boussinesq approximations, the speed reached by free convection could  
215 be calculated. Indeed, for a chimney with a height of about 1km, deviation from the  
216 exact solution given by the Boussinesq approximation is negligible and the error is  
217 trivial [51].

218 Even though it is reported that the output power grows with the square of the  
219 chimney height [46] as a result of the increased mass flow rate, approval for a large  
220 chimney height are not always available so this claim has yet to be fully verified. In  
221 addition, there are still constructional limitations, and other natural hazards are  
222 inevitable when making such a large chimney. Moreover, aero thermal  
223 characteristics of flow, inside and over different chimney shapes, could strongly  
224 affect power output. Backflow concerns should also be considered.

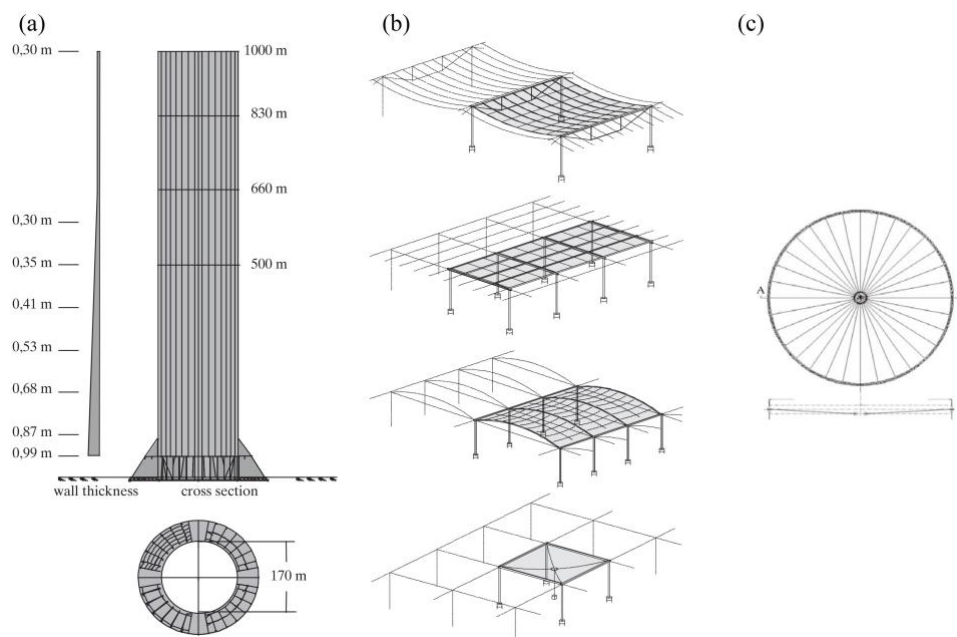
225 Nonetheless, many configurations have been proposed to cope with this issue,  
226 although a large chimney is still vulnerable to a harsh wind storm and other natural  
227 forces like seismic activities [52] especially with super-tall chimneys and for non-  
228 urban areas which have a greater interest in their operation [53-55]. Wang and Fan

229 summarized 739 types of high chimneys failure cases and compared effective  
230 factors in observed failures in a complete review [56].

231 Schlaich [57] considered the danger of buckling as the reason for the height  
232 limitation of natural draught cooling towers which was about 200 m. He suggested  
233 stiffening spoked wheels as a countermeasure.

234 Many layouts have been proposed in respect of structural choices to deal with  
235 possible structural failure, but the best choices of material still proved to be  
236 reinforcement concrete, guyed tubes made from corrugated metal sheet, and cable-net  
237 designs with cladding or membranes which is an appropriate choice for less  
238 developed countries [50, 58, 59]. Constructional consideration for the chimney wall  
239 thickness would suggest decreasing wall thickness from about 1 m just above the  
240 support on radial walls to 30 cm halfway up, which then remaining constant all the  
241 way to the top, stiffen at several levels with cables arranged like spoked wheels  
242 within the tower to counteract over-toppling caused by wind suction in flanks and  
243 the use of these thinner walls [59]. Structural configurations are illustrated in Fig.  
244 6.

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Fig. 6. Structural configurations: (a) Wall thickness variation of a 1000 m height and 170 m diameter tower; (b) Collector design options; and (c) Spoked wheels, the spokes are made of vertical steel slats [59]

252 However, a limiting factor in an increased chimney height is the fall in rising air  
253 temperature due to heat loss and its decreased velocity due to flow loss, with  
254 subsequent reduction in buoyancy. Accordingly, the greater flow loss and lower  
255 buoyancy effects with higher chimneys limits chimney height for optimized power  
256 output. Demonstrated by Zhou et al [45] for the first time, they emphasized that  
257 there is a maximum height for the chimney, and this that increases with the collector



258 radius. To overcome this limitation researchers then started looking at optimising  
259 other parameters [60]. For example, it was shown that an increase in the collector  
260 area could compensate for a lack of chimney height [50]. Cottam et al. [46]  
261 suggested a linear relationship between power output and collector radius the same  
262 as the chimney radius. Schlaich [57] claimed the same output may result from a  
263 large chimney with a small collector roof area and vice versa – although this is not  
264 strictly a linear correlation (between power output and collector area times tower  
265 height) because of collector friction losses [50] – and he also implied there is no  
266 optimum physical size for solar Chimneys. Additionally, to decide the optimum  
267 dimensions the specific construction costs of each item must be known.

268 From a building cost viewpoint, Wolfgang Schielcl suggested that operating a large  
269 chimney is much cheaper than operating many small ones [58] even though Cottam  
270 [46] prefer several smaller collector with the size of 3000m over a very large one.  
271 More recently, other investigations to cope with chimney difficulties have been  
272 undertaken but the discussion still continues. For example, to evaluate the effect of  
273 diameter, a chimney ‘slenderness’ ratio parameter has been defined which is the  
274 ratio of chimney height to chimney diameter. Normally slenderness ratio could vary  
275 from 5 to 12 influenced by various reasons. However, Petrorius [61] reported an  
276 optimal slenderness of 5-6, considering the risk of cold air inflow. There is also a  
277 critical value of 6-8 and it is claimed that the diameter has a prominent effect under  
278 this value [62]. Considering the influence of winds, Kashiwa [56] suggested the  
279 slenderness ratio of 12 and mentioned that wind variation could have a great  
280 influence in the updraft.

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## 282 2.2 Collector

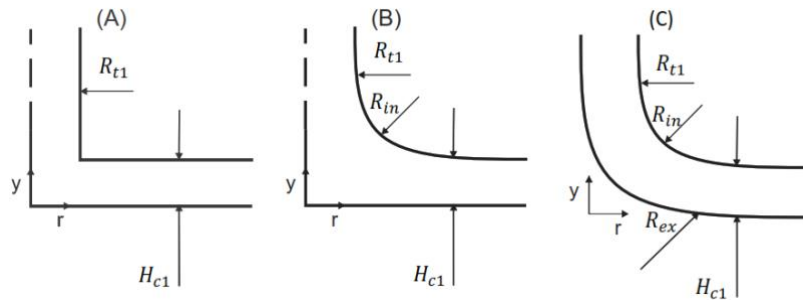
283 One of the major parts of the SCPP, the collector plays the role of a heat exchanger  
284 within the system, in that the collector converts solar radiation into thermal energy  
285 utilizing the greenhouse effect. The thermal energy of a heat absorber first warms  
286 the air and then the thermal energy of this heated air is converted to kinetic energy  
287 due to its buoyancy effect. A transparent roof, column structure and support matrix  
288 are all major parts of the collector, and the principal mechanism occurs when  
289 irradiance through the transparent canopy hits the absorber section. The transparent  
290 canopy, which is often plastic or glass, is then not able to pass any infrared radiation  
291 emitted from the absorber back to atmosphere, but instead the absorber heats air  
292 which is then exploited by the overall process. Although, as has been mentioned,  
293 that output power is directly proportional to the collector radius, similar to chimney  
294 height, the collector area is limited. Guo et al reported a maximum collector area  
295 for the Spanish prototype through a MATLAB program [49]. Collector inlet height,  
296 collector inclined angle and collector profile shape are the major parameters of this  
297 study. Despite great effort from many researchers, the issues governing an optimal  
298 collector taking in to account all parameters, remains, at this time, still unsolved –  
299 the same as with the chimney. Because optimal dimensions must include economic  
300 factors, many researchers have pragmatically employed a multi-objective

301 optimization approach [63, 64], from which various researchers have proposed  
 302 alternative ideas to improve collector efficiency and for harvesting more power.

303

304 **2.3 PCU**

305 Finally, the Power conversion unit (PCU) consists of one (or more) turbo-  
 306 generator(s) (turbine coupled with generator), the output from which depends on the  
 307 air mass flow rate being fed to it from a horizontal to vertical transition section  
 308 (HTVTS) between the collector and the chimney. With some designs, inlet vanes  
 309 have been used to redirect and guide the flow through the turbine, but the PCU may  
 310 also include a diffuser located behind the turbine for the single turbine  
 311 configuration. Bernardes et al. [65] presented a series of possible configurations for  
 312 the HTVTS based on three basic geometric configurations which are given in Fig.  
 313 7, simulating thermodynamic behavior using natural laminar convection. The  
 314 concern in the power conversion unit being the recirculation of air flow caused by  
 315 unsuitable configuration choices. Bernardes et al. stated that mass flow rates for  
 316 rising air are greater for a conic SCPP, and Yan et al. [30] considered that straight  
 317 junctions induced the lowest mass flows due to flow recirculation. Later Muller [66]  
 318 found a 43 percent loss reduction in multiple horizontal axes with guiding cones.  
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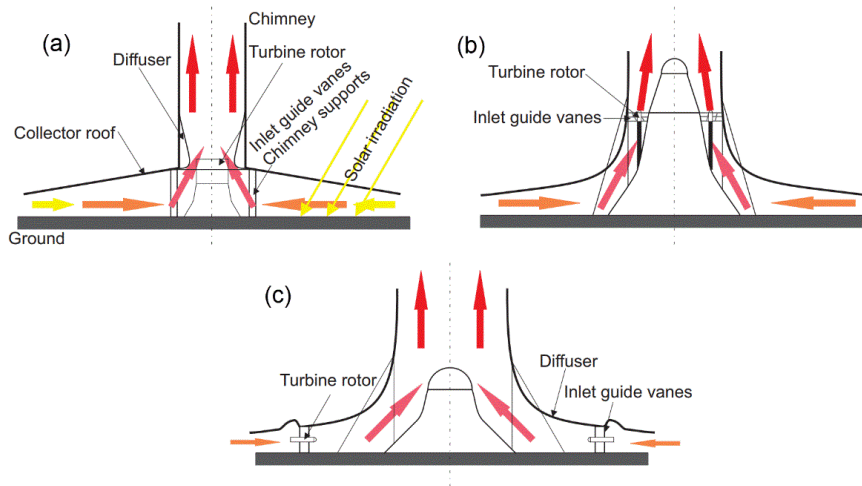
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Fig. 7. Structural configurations of horizontal to vertical transition section (HTVTS) [65].

322 With regards to the siting of the turbine, some layouts have been proposed.  
 323 Pasumarthi and Sherif [14] suggested installation at the top of SCPP whereas in  
 324 most literature, turbines are located in the base because of installation and  
 325 maintenance difficulties, especially for super large SCPPs. Bonelle [23] observed a  
 326 relatively negative pressure in SC for in-base installation and relatively positive  
 327 pressure for in-top installation, explained by the fact that static pressure must drop  
 328 from upstream to downstream. Further, many arrangements have been proposed for  
 329 the actual turbine configuration – a single vertical axis, multiple vertical axes and  
 330 multiple horizontal axes that could contain inlet guide vanes (IGVs) or not (all as  
 331 shown in Fig. 8). Schwarz and Knauss [67] designed a single vertical axis for SC  
 332 and Gannon and von Backstrom [68] utilized supporting structures for IGVs. Bilgen  
 333 [69] suggested one pair of counter-rotating rotors as an alternative turbine layout.  
 334 In fact, the proper choice depends on the solar chimney size. In small solar  
 335 chimneys, multiple turbines may not be a good selection, whereas for super large  
 336 systems this could reduce manufacturing costs and maintenance challenges. In the  
 337 power conversion unit junction, the shape of the collector into the chimney also has

338 a significant effect on the thermo-hydrodynamic field quality. Chergui et al. [70]  
 339 investigated different junction shapes and their resultant junction with the diffuser  
 340 had a higher mass flow rate in comparison to the curved junction and the straight  
 341 junction.  
 342



343  
 344 Fig. 8. Configuration of turbine installation: (a) single vertical axis; (b) multiple vertical axis and;  
 345 (c) multiple horizontal axis[1]

346  
 347 Ayadi et al. [71, 72] investigated the effect of turbine diameter and the number of  
 348 blades and reported that turbines with the largest diameter and the lowest number  
 349 of blades are the best option for small scale SCPPs. Kasaeian [73] conducted a 3D  
 350 simulation of large scale SCPP considering turbine blades. However, contrary to  
 351 Ayadi's experimental setup, Kasaeian reported that a turbine with 5 blades, presents  
 352 more power output than that with 3 blades, whilst 3-bladed turbines provided a  
 353 higher mass flow rate. More research is needed in this subject to determine an  
 354 optimal turbine configuration.  
 355

### 356 3. Innovative Ideas and Alternative Strategies

357 Despite the clean and eco-friendly energy production from SCPP systems, their  
 358 disadvantages of noticeably low efficiency and high construction costs are  
 359 considerable. To reduce these problems, researchers have focused on finding the  
 360 optimum configuration and alternative choices to improve SCPPs efficiency.  
 361 Optimizing major parameters, studying different configurations, utilizing different  
 362 mechanisms such as thermal fins or accessories like intensifiers, and combining  
 363 these with alternative systems such as water desalination – have been widely studied  
 364 with a view to improving system power output and providing a reasonably efficient  
 365 economic system.  
 366

#### 367 3.1 Different Geometry Configuration

368 To deal with low efficiency the first and simplest option is to optimize the major  
 369 components and make changes in their basic configuration. This is simple and  
 370 incurs lower costs in comparison to integrating systems or involving different

371 accessories. Numerous researchers have focused on applying different  
372 configurations and optimizing related parameters in an effort to improve SCPP  
373 efficiency.

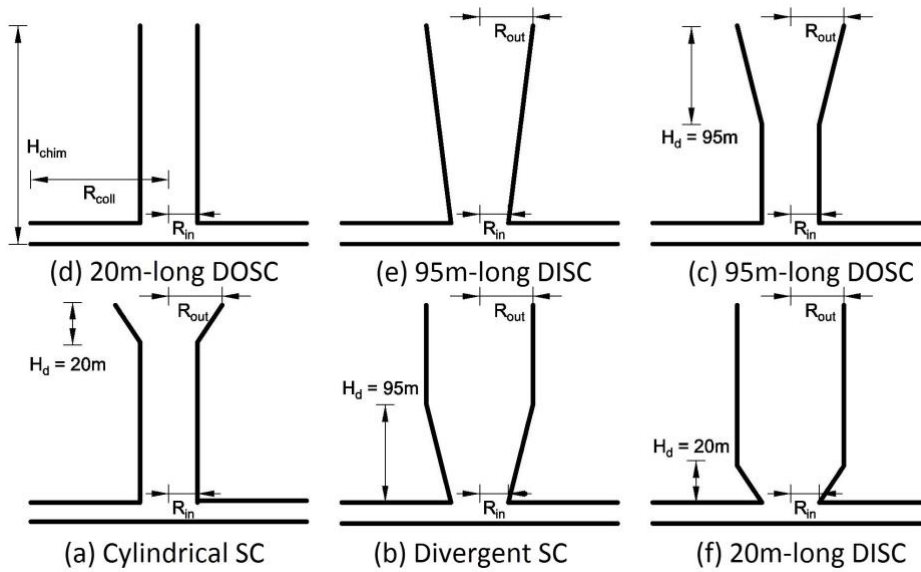
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### 375 3.1.1 Chimney

376 One focus has been an investigation of different chimney configurations, and these  
377 include changing the cylindrical shape of the chimney or presenting novel layouts.  
378 Ming et al. [74] studied the effect of divergent and convergent chimney angle and  
379 chimney height to allow a better understanding of different shape parameters. They  
380 compared results obtained against a reference case including a cylindrical chimney  
381 shape. With the divergent type of chimney, the diameter increases slightly with  
382 height, and this affects the low static pressure at the chimney inlet and consequently  
383 has a significant effect on the air velocity value which promotes greater airflow  
384 inside the system. Okada [75] et al. suggested a diffuser type chimney to increase  
385 the power generation in the system. The result of their CFD analysis in large scale  
386 plants revealed that a focused airflow in the throat increased power generation and  
387 the flow speed throughout the chimney, but especially at the bottom – a precept  
388 which was further studied on an indoor scale solar chimney. They concluded that a  
389 divergence angle of 4 degrees increased the power output by about 3 times that of  
390 a straight chimney. Jameei et al. [76] assessed 15 types of chimney walls based on  
391 a three-step procedure from convergent-divergent form to circular concave-convex  
392 and parabolic curve. The results showed about a 50 percent increase in velocity for  
393 the parabolic curve. Koonsrisuk et al. [77] investigated both the effects of collector  
394 slope and chimney convergence and divergence angle. Patel et al. [78] also  
395 performed a series of twelve case studies to investigate the effect of chimney  
396 divergent angle and collector height. Ohya [79] et al performed an indoor laboratory  
397 experiment analysis and showed power output for diffuser-type towers can be  
398 increased by 4 to 5 times in comparison to conventional towers, but also showed a  
399 dependence on air temperature increments inside the collector. Nasraoui et al. [80]  
400 found that the chimney divergence angle has an optimum value depending on  
401 chimney height. Their findings revealed that the optimum divergence angle to solar  
402 chimney scale relationship is nonlinear, and that the value of the divergence angle  
403 decreases by increasing the solar scale and hence for commercial SCPPs the  
404 optimum divergence angle would be the more suitable choice.

405 Hu et al. (Fig. 9 [81]) carried out several case simulations to discover the best shape  
406 for the diffuser type chimneys, and they reported that diffuser type chimneys could  
407 handle more than 13 times the power output than simple cylindrical solar chimneys.  
408 They showed different parameters of AR (area ratio) and  $H_d/R_{in}$  (where  $H_d$  is  
409 divergent section and  $R_{in}$  is the start of the divergent section) could affect the  
410 aerodynamic characteristics of flow inside the chimney and expansion loss, both of  
411 which directly affected performance. Comparing simulation results revealed that a  
412 divergent solar chimney configuration which has the largest  $H_d/R_{in}$  is best option  
413 for power generation, DISC is second-best option and DOSC is the last. Hu  
414 proposed a controlling approach for the design of a solar chimney containing a

415 variable diffuser outlet. The concept being that the user could change the area of the  
 416 outlet and so adjust the fluctuating power output.  
 417



418  
 419 Fig. 9. Examination of three different diffuser type carried out by Hu[81].

420 Hu et al. [82] studied a wide range of AR changes of the tower from 1 to 32 and  
 421 emphasized that power output improvement is dependent on chimney height.  
 422 However, the situation is not that simple. Divergent chimneys can act like a diffuser,  
 423 and this has a concerning impact on flow characteristics. Formation of large eddies  
 424 near the wall in a diffuser represents the onset of stall phenomena, which could then  
 425 lead to flow blockage and unwanted backflow. In the Xu Zhouet al. [83]  
 426 investigation, with small outlet to inlet AR cases, flow goes up normally without  
 427 any backflow, but the stall phenomena was observed for outlet to inlet ARs larger  
 428 than 11.9. For even larger divergence angles, backflow occurs over a larger  
 429 proportion of the flow area and boundary layer separation occurs lower down within  
 430 the chimney. This backflow brings ambient air into the chimney, which then reduces  
 431 the average temperature and leads to a large reduction in buoyancy and pressure  
 432 potential. Further chimney configurations, such as convergent chimneys and  
 433 opposing (convergent-divergent chimney) designs have examined by Bouabidi et  
 434 al. [84]. Their results showed that the air velocity values are affected differently by  
 435 each configuration. For example, the divergent chimney revealed the best results in  
 436 this study, whereas the convergent configuration degraded maximum velocity, and  
 437 opposing chimney only enhanced the velocity compared to the standard case.  
 438 Chimney divergence and convergence investigations are summarized in Table 1.

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Table 1. Chimney divergence and convergence investigations

Investigated divergence and convergence angle <sup>†</sup> (DA), (CA) and area ratio (AR)	Optimal range	Chimney height(m)	Power output <sup>‡</sup> equalized by the power of reference case (AR=1 OR DA=0)	ref
AR (div):1.56,2.25 AR (con):0.25,0.56	-	800	AR (div):1.06,1.07 AR (con):0.8,0.95	Ming[74]
AR (div):1,2,4,8,16,32 AR (con):0.25,0.5,0.75	AR (div):16	100	AR (div):1,4,27,18.49,69.07, 179.16,120 AR (con):0.02,0.09,0.19	Koonsrisuk et al.[77]
DA:1,2,3	2	10	7,10,9.8	Patel et al.[78]
AR (div):3.09,12.39,49.56	12.39	10	7.9,13.6,11.2W (absolute power)	Vieira et al.[85]
DA:4	-	0.4	3	Okada[75]
DA:2.4,4,7 CA:4,7	DA:2.4	-	DA:4.2,3.9,2.8 CA:0.8,0.4	Chergui[70]
DA:0,2,4 and 6	4	2	11.8,31.93,52.5, 46.74 mW.(absolute power)	Ohyaet al.[79]
AR (div):1,4,10,22,32	10	200	1,8.5,13.5,10,6.6	Hu et al.[82]
AR (div):3.94,8.76,11.83	8.76	194.6	7.58,11.9,7.2	Xu et al.[83]
DA:0,1,2,3	1	195	34,70,64,49 kw (absolute power)	Aakash Hassan et al.[86]
AR (div):4,6,8,10,12	10	195	440,604,678,702,695kw (absolute power)	Hu et al.[81]
AR (div):4,9,16,25,36 AR (con):0.25	AR (div):16	12.3	AR (div):15.3,24.3,26.1,19.5, 15.3 AR (con):0.2	Lebbi et al.[87]
DA:2,3,6,9	3	100	66,69.3,61,44.5	Nasraouiet al.[80]
DA:1,2,3,4,5	2	7	0.63,1.07,1.04,0.95,0.86 (W power)	Pritam et al.[88]

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Further researchers have investigated different practical configurations for the updraft tower manipulating the chimney profile. Nasraoui et al.[89, 90] proposed a hyperbolic-shaped profile for the chimney and the results suggested an enhanced

<sup>†</sup> degree

<sup>‡</sup> The extracted values from diagrams and tables and in case of velocity report values obtained from turbine model power output  $P = 0.5 \cdot \rho \cdot A \cdot u^3$

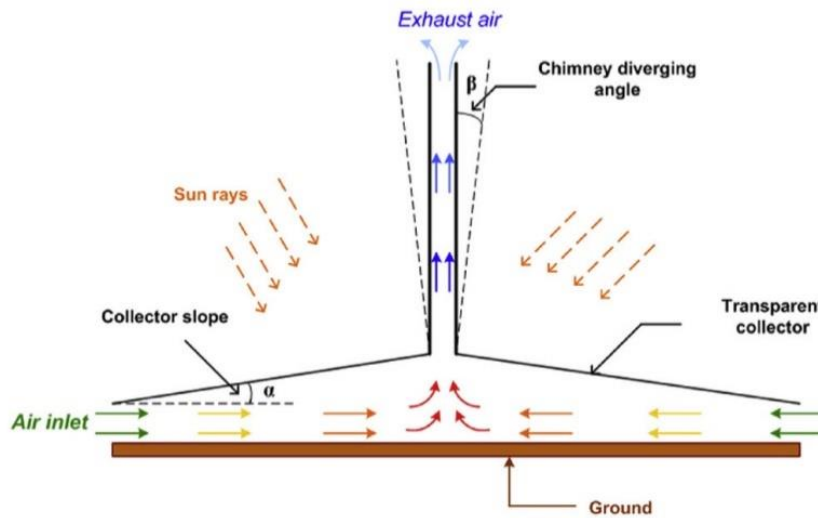
451 power output in most cases. As before, the power rise suggests an optimum range,  
452 but these are more effective for larger outlet to inlet diameter ratios, showing almost  
453 constant chimney efficiency changes with very large diameter ratio, whilst with the  
454 conical section, degradation was higher in large values. In comparison against  
455 conventional conical chimneys, the hyperbolic chimney showed a 45 percent  
456 enhancement with a diameter ratio of 8. This comes because a conical chimney  
457 increases power by 250 percent compared with a straight chimney with the same  
458 diameter ratio. The conclusion is that a diverging chimney, with different shapes  
459 might then form a key parameter in enhancing the performance of SCPPs and so  
460 present an attractive solution for low-efficiency SCPPs.

461 A further idea of utilizing air flowing into a low static region was suggested by  
462 Okada et al.[75, 91]. They suggested utilizing a ‘brim’ that acts as a vortex generator  
463 and so provides a low-pressure region in the chimney outlet, and this idea, exploited  
464 in the wind-lens, has resulted in a two - threefold increase in power output [92].  
465

### 466 3.1.2 Collector

467 The collector is the second major component of a SCPPs and has a significant effect  
468 on the performance of the system. Early researchers have therefore investigated the  
469 impact of the major collector parameters in order to better understand the effect of  
470 these [93]. And whilst the dimensions of the collector radius and the chimney height  
471 are considered factors directly affecting power output, many investigations have  
472 suggested that collector radius could have a greater significance on performance and  
473 consequently, any increase in chimney size is more effective with large diameter  
474 collectors [94, 95]. Ming et al. [74] studied the effect of collector height in addition  
475 to the collector radius, to better understand the different shape parameters, and  
476 although decreasing the collector height led to an increased power output, collector  
477 height showed of less importance compared to the other major parameters such as  
478 chimney height, collector radius and chimney diameter [85, 94]. Sandeep et al. [78]  
479 stated there is an optimum value for the inlet opening height and that, in fact, a very  
480 low of inlet opening height could end up reducing the power output because of a  
481 lower mass flow rate. Flat collectors cause pressure drop due to a cross-sectional  
482 restriction of flow near the chimney, but sloped collectors are a promising solution.  
483 Hence the slope of the collector could play an important role related to the quality  
484 of hydrodynamic flow inside the chimney, the same as chimney divergence and  
485 convergence angle as discussed earlier (Fig. 10). Hakim Semai et al.[96] reported  
486 an entropy reduction in converging SCPP. Sun et al. [97] also investigated the effect  
487 of inclined angle vs. power output and other flow characteristics for the collector.  
488 They observed vortices appearing and increasing inside the collector, especially  
489 near the chimney position, with positive inclination angles ( $\beta$ ). Aakash [86] and  
490 Ayadi [98] also studied the effect of the collector slope, and reported that a negative  
491 inclination angle leads to a higher velocity and hence a higher power output as a  
492 result. Ayadi suggested a 125 percent increase in velocity for a variation of 2.5  
493 degrees in inclination angle. Aakash also suggested that increasing the inclination  
494 angle could result in a higher mass flow rate and larger air velocity. Although very

495 large angles – about 6 degrees – could lead to air recirculation within the collector  
 496 due to a density gradient and hence cause air blocking as a result.

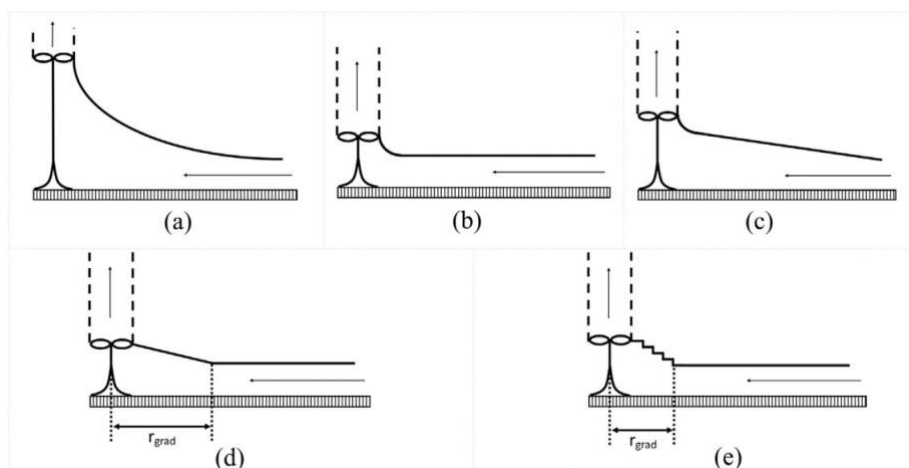


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Fig. 10. Chimney diverging angle  $\alpha$  and collector inclined angle  $\beta$ [86]

499 Sandeep et al. [78] investigated the influence of both collector outlet height and  
 500 collector outlet diameter. Their results for the collector output height correlates well  
 501 with other investigations and indicates that a large collector output height could  
 502 limit growth in power output. They also observed a strong effect regarding the  
 503 collector outlet diameter and claimed that with a large value of collector outlet  
 504 diameter, a larger quantity of air is able to enter the chimney with a reduced flow  
 505 resistance producing greater power. Kebabsa et al. [99] investigated different  
 506 collector entrance configuration parameters, including sloping distance and angle.  
 507 They proposed even better result in comparison to normally inclined collectors, with  
 508 up to 13 percent more power output than a conventional inclined collector. Cottam  
 509 et al. (Fig. 11[100]) investigated a series of canopy profiles for the optimal canopy  
 510 layout and power output.

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Fig. 11. Collector profiles: (a) exponential; (b) flat; (c) constant-gradient sloped; (d) segmented; and (e) segmented & stepped [100]



515 They compared the results for an exponential profile, a flat profile, a segmented  
516 profile, and a segmented with stepped profile, and reported that collector outlet  
517 height is an important parameter and could help control the pressure drop in the  
518 collector to chimney transition section. The adequacy of cross-sectional area in the  
519 transition section is a known determinative parameter for pressure loss. And even  
520 though an exponential profile has proved to be the best choice, Cottam et al. showed  
521 that a segmented collector profile could almost reach the performance of an  
522 exponential profile, but with a simpler design and lower cost. They also suggested  
523 a stepped segmented profile for good power output to construction cost balance.  
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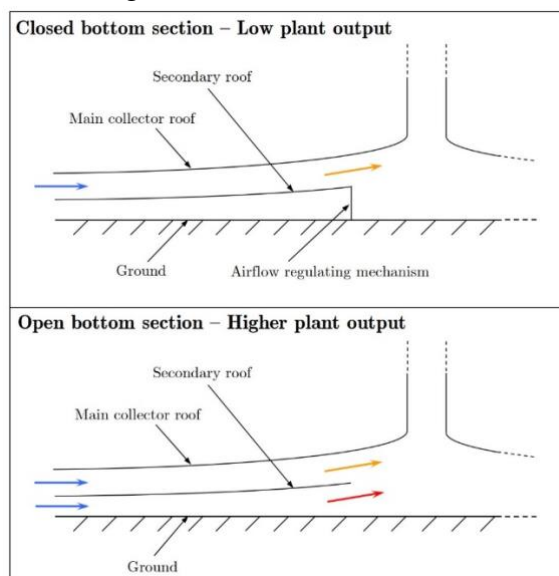
### 525 3.2 Accessories and Integrated Apparatus (Auxiliary Techniques)

526  
527 Many researchers have tried adding accessories to help improve the performance of  
528 SCPP systems; for example, supplementary apertures aim to enhance either the heat  
529 transfer or the flow field inside the solar chimney. Relating to this, Hosseini et al.  
530 [101] performed a numerical simulation to study the effects of longitudinal  
531 rectangular fins on solar chimneys, both in continuous and discontinuous fins.  
532 Comparing these to a flat absorber, Hosseini declared that, with appropriate  
533 interruption gaps, discontinuous fins could enhance the performance of solar  
534 chimneys over and above that produced from continuous fins. They showed that  
535 despite the dependence of efficiency to absorber area, the efficiency could still be  
536 improved by increasing the depth ratio of fins because the heat transfer area, and  
537 hence the net heat transferred to the airflow will be increased as a result. They also  
538 noted that disturbance and reformation of the boundary layer could lead to an  
539 improvement in the heat transfer coefficient but in the appropriate gap due to the  
540 effect declining in the absorber area. They also reported that increasing the number  
541 of fins would improve thermal performance. Hosseini et al. [102] also compared  
542 different shapes of fin, such as triangular, elliptical and rectangular in natural  
543 convection solar air heaters and declared that the thermal efficiency of a solar air  
544 heater with rectangular fins is 5.5% higher than those containing elliptical and  
545 triangular fins, and also that the thermal enhancement is strictly related to an  
546 increase in solar radiation and reduction of ambient temperature.

547 Following a different approach, Shabahang and Ghazikhani [103] studied the effect  
548 of passive flow control devices on solar chimneys. They implemented different  
549 obstacles shapes such as half-circle, rectangular and triangular obstacle shapes to  
550 alter the flow field and so improve inflow mixing using secondary flows and vortex  
551 generation. They also considered boundary layer agitation and fluid mixing as a  
552 contributor to enhancing the Nusselt number. They study observed improvements  
553 in all cases, but an obstacle with a triangular profile produced the greater thermal  
554 performance enhancement, because the flow pattern was not blocked, but rather was  
555 guided toward the chimney. Their passive control resulted in an increasing velocity  
556 rate and consequent energy output improvement of up to 41.2%. Further, Fallah et  
557 al. [104] evaluated the effect of artificial roughness on the ground as the air passes  
558 through the collector. They considered an optimal location for this artificial

559 roughness, which reduces air velocity but improves heat transfer compared to  
560 without-roughness collector cases. Hence, either from natural roughness (like the  
561 ground surface) or from the artificial roughness of a form of energy storage system,  
562 roughness has proved to make a valuable impact on solar chimney system  
563 performance. That being said, it has been suggested that installing artificial  
564 roughness near the collector entrance has no significant impact. So, to cope with the  
565 difficulty and limitations of a large installation field and higher collector efficiency,  
566 many researchers have applied different configurations and technologies to resolve  
567 this problem.

568 Pretorius (Fig. 12)[61] introduced a secondary roof integrated with an airflow  
569 regulation mechanism. In the Pretorius concept, the bottom section remains closed  
570 when less power is needed, and the system is in the ground energy-storing mode.  
571 At other times, the bottom section is opened, and flowing air captures the energy  
572 previously stored within the ground.

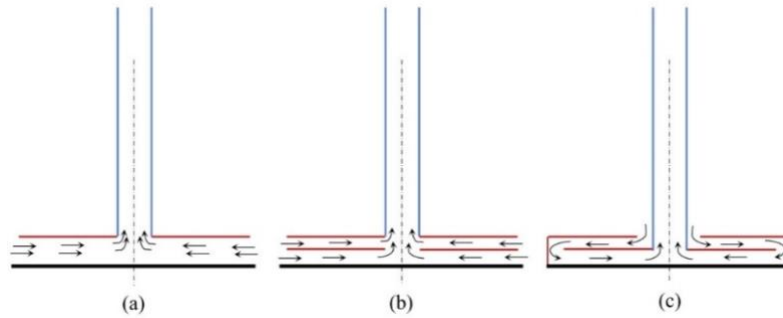


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Fig. 12. Introduced a secondary roof collector by Pretorius[61]

575 Nasraoui et al. (Fig. 13) [105] suggested a novel collector design utilizing two flow  
576 paths. They developed a comparison between (a) standard configuration, (b) parallel  
577 in-flow channels and (c) counter in-flow channels. The results show greater  
578 improvement for the counter-flow case, but improvements occurring in both cases  
579 with parallel and counter-flow configurations to maximum velocity and temperature  
580 rise.

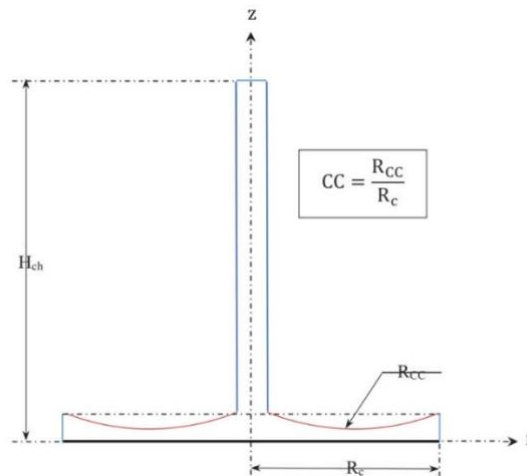
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Fig. 13. Different configuration of novel collector design (a) standard collector; (b) double roofs collector with the parallel flow; (c) double roofs collector with counter flow [105]

585 Nasraoui et al. [106] predicted the effect of concavity in the collector by defining  
586 the function of the concavity ratio as the curvature radius of concavity in the  
587 collector roof divided by collector radius. They proposed that a concave collector  
588 could enhance the performance of the system and produce more power by increasing  
589 the concavity ratio of the collector, by which, understandably, the velocity inside  
590 the collector is increased (Fig. 14).



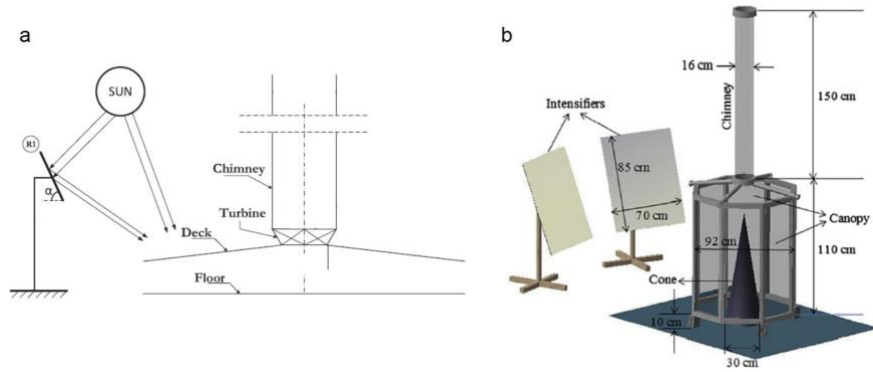
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Fig. 14. Concavity ratio of a collector [106]

593 Rezaei and Imani [107] carried out a novel investigation on a small scale solar  
594 chimney setup utilizing intensifiers to increase heat flux on the collector and hence  
595 increase incoming irradiance. The layout is depicted in Fig. 15a. They also located  
596 an air tank **downside** the system to increase the absorption of solar radiation  
597 reflected by the intensifiers. In addition, a mechanical assembly was designed to  
598 allow the mirrors to traverse the orbital path in order to track all-day sun movement.  
599 They showed that, by using this apparatus, air velocity within can be chimney  
600 increased and this leads to greater power output. They recorded a maximum air  
601 velocity of 5.12 meter per second – which was remarkable for the test plant size –  
602 showing an approximate twofold increase compared to a without-intensifier  
603 prototype [108].

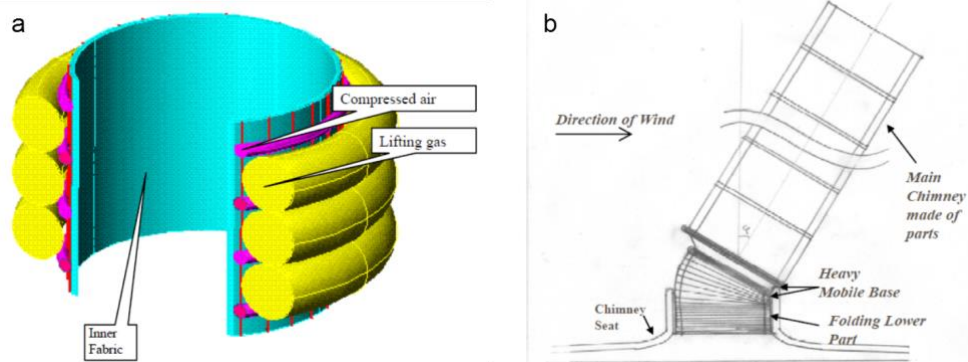
604 Faisal Hussain et al. [109] formulated an exergy and energy balance equation for  
605 the SCPP and observed maximum exergy destruction, and consequently the highest  
606 improvement potential, happens at the floor. They therefore proposed a new design

607 of solar chimney with enhanced incident solar radiation aided with reflectors  
 608 depicted in Fig. 15b. As a result, the efficiency gain was up to 22 percent compared  
 609 to a conventional solar chimney, and all resulting from an almost ten percent  
 610 increase in floor temperature.



611 Fig. 15. A schematic of SCPP setup aided with intensifiers (a) Rezaei and Imani[107]; (b) Hussain  
 612 and Al-Sulaiman[109]  
 613

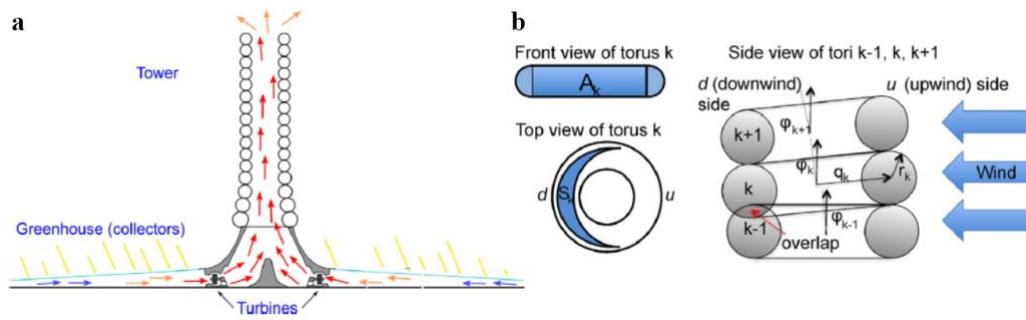
614 Many authorities have reported that a main obstacle to designing very tall chimneys  
 615 is that caused by the crosswind forces, and that these induce transversal stresses to  
 616 the construction which need to be resolved for before successful commercializing.  
 617 The design considerations have therefore caused a lot of difficulties for such rigid  
 618 tall structures and produced many patents applications to cope with the problem.  
 619 One such innovative configuration for a tall chimney has been proposed by  
 620 Papageorgiou [110-117] (Fig. 16). In which, in order to reduce the chimney  
 621 construction costs and resolve the difficulties, he utilized the merits of the airlifting  
 622 force from an inflated fabric structure instead of a heavy rigid concrete body. The  
 623 floating solar chimney (FSC) design therefore consists of an inner polyester fabric,  
 624 associated with a twisted tube around it, which is then filled with lighter-than-air  
 625 gas. He suggested either He or NH<sub>3</sub> as the filling gas to provide the role of the lifting  
 626 force and proposed that such a floating solar chimney section could cope when  
 627 combined to form super tall towers and be constructed for heights of up to 3000  
 628 meters. The folding lower part are also designed to bend freely against external  
 629 winds and so can easily handle crosswind threats. It should also be noted that  
 630 chimney height might be limited by annual average wind speeds and the potential  
 631 deviation angle caused by prevailing crosswinds.  
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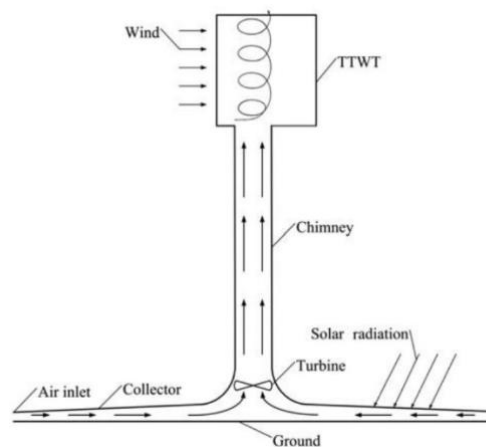
634 Fig. 16.(a) structure of FSC with over-pressed air tubes that retain its cylindrical shape and lifting  
 635 gas tube to make a tall lighter than air cylinder[118]; and(b)configuration of FSC with folding  
 636 lower part and floating chimney seat to deal with crosswind[113]

637 Zhouet al. [119] and Maghrebiet al. [120] also carried out an investigation into their  
 638 economic aspects and estimated cosst. Maghrebi showed that these floating power  
 639 plants are able to be constructed at large scale of up to 200 MW of electricity, with  
 640 an annual capacity of 641 GW. In comparison, Putkaradze et al [121] suggested an  
 641 innovative self-supporting, free-standing and flexible solar tower constructed with  
 642 air-filled stacks as a replacement for a vulnerable tall steel chimney, as illustrated  
 643 in Fig. 17. In their model, the chimney no longer has a straight cylinder geometry  
 644 but can deform, and such deformations are not concentrated just in the base of the  
 645 chimney – unlike the former design.



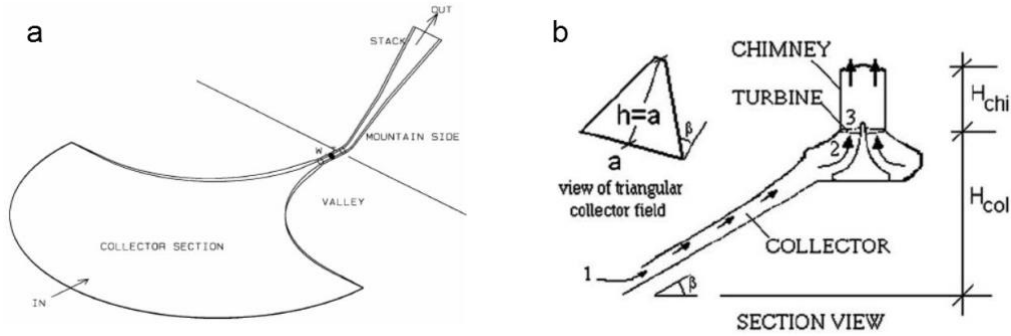
646 Fig. 17.Schematic of (a) solar chimney composed of toroidal bladders;(b) wind deformation  
 647 mechanism shown[121]  
 648

649 An third interesting and novel concept was proposed by Li et al .[122], in which  
 650 they suggested a combination of tornado type wind tower combined with a  
 651 conventional SSCP. In this concept, the tornado type wind tower is positioned at the  
 652 top of the chimney to exploit deficits of pressure and thereby increase the updraft  
 653 driving forces. Li et al. showed the proposed prototype could decrease solar  
 654 chimney power plant height by almost two times with a wind velocity of 15m/s. In  
 655 conclusion, the hydrodynamic effect of wind speed enhances SSCP efficiency and  
 656 could play a key role in solving one of the problems of tall chimney construction.  
 657 The prototype is presented in Fig. 18.



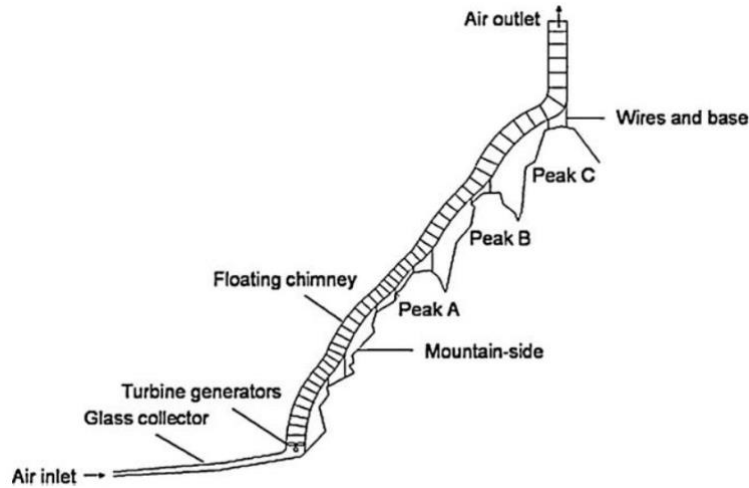
658 Fig. 18. Combined tornado type wind tower and SSCP concept [122]  
 659

660 Yet another suggested design to reduce chimney construction problems has been  
 661 mentioned in Serag-Eldin's literature (Fig. 19a [123]). The goal being to exploit the  
 662 height of a mountain as a replacement for very tall, vertical chimneys, and all with  
 663 with no moving parts. With Serag-Eldin's proposal, the chimney utilizes the slope  
 664 of the mountain, and runs up a ground-laid duct whilst the collector spreads over  
 665 valleys. In fact, with such a design, the collector is limited to a semicircular  
 666 geometry.  
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668 Fig. 19. Sketch of (a) proposed solar chimney design for mountainous regions [123]; and (b) solar  
 669 chimney systems in a sloped surface at high latitudes [13].  
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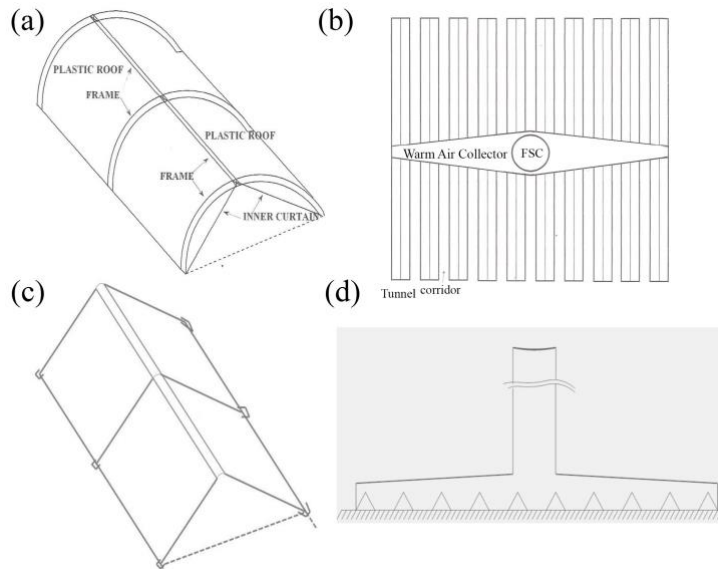
671 Additionally, a sloped solar chimney has been produced by Bilgens (Fig. 19b [13])  
 672 for use in high latitudes, the main characteristic of this concept being a sloped  
 673 triangular-section closed side collector which follows the line of the mountain or  
 674 other natural slope. It was shown that this feature could then accept a solar chimney  
 675 with height up to half the height of the collector [124]. Jing et al. investigated the  
 676 best slope gradient in this area [125]. Some researchers [38, 126-129] also carried  
 677 out performance analysis of these systems, comparing sloped solar chimneys, with  
 678 conventional solar chimneys in China, Iran and other locations. Kalash et al. [130]  
 679 investigated the temperature field of a sloped solar updraft power plant  
 680 experimentally. Xinping Zhou [131] studied the best curved-profile fit for a  
 681 mountain profile. He compared linear sloped collectors with a two-segment sloped  
 682 collector and showed that the optimal sloping angle is lower than the local latitude.  
 683 A further novel concept in this area has been presented by Zhou et al. [132]. Zhou  
 684 [133] who suggested that a Solar Thermal Power Plant with Floating Chimney,  
 685 rigidly mounted onto a Mountainside alleviate some of the difficulties of floating  
 686 SCPPs, and make a smoother chimney line in mountain regions. The concept is  
 687 shown in Fig. 20.



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Fig. 20. Floating SCPP rigidly-mounted for a mountainous region, schematic [133]

690 Papageorgiou (Fig. 21) [118, 134] proposed a modular Solar Collector for the solar  
691 chimney technology as a lower-cost alternative of the usual circular greenhouse  
692 pattern. The greenhouse will be made from a series of parallel reverse Vs or U-  
693 section glass panel tunnels leading to the entrance of the FSC. The modular solar  
694 collector diminishing on-site work and so lowering construction costs. It also could  
695 reduce dust problems and subsequent mud formation [135] which arise from a  
696 condensate film on the surface.



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Fig. 21. Structural configurations of modular Solar Collector: (a) U type; (b) top view; (c) V type; and (d) front view [118]

701 Bonnelle [136] proposed a fresh concept to reduce turbulent friction inside the  
702 collector by utilising a series of branching ribs as shown in Fig. 22. Bonnell's design  
703 for the collector entrance, possessing a larger area than a conventional collector,  
704 leads to lower velocities and hence helps reduce turbulent friction losses. At the  
705 same time, the collector roof can be installed with a lower height, offering the  
706 opportunity for reducing costs.

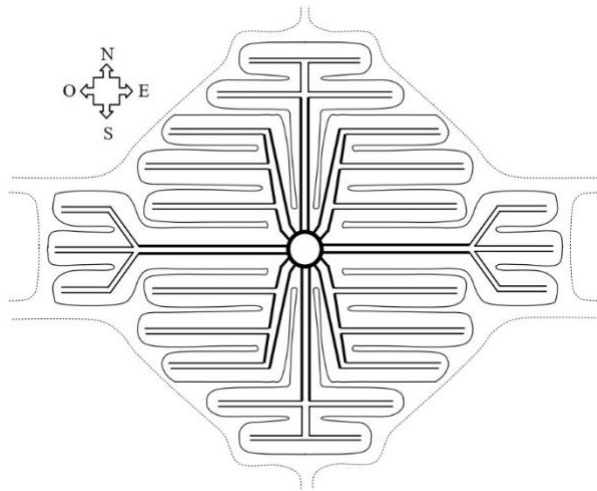


Fig. 22. Collector configuration with branching ribs [136]

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709 Papageorgiou [137] introduced a new concept to replace the gigantic chimney cross-  
710 section turbine by a series of controlled air opening in the collector inlet.

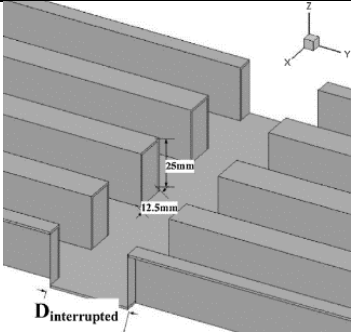
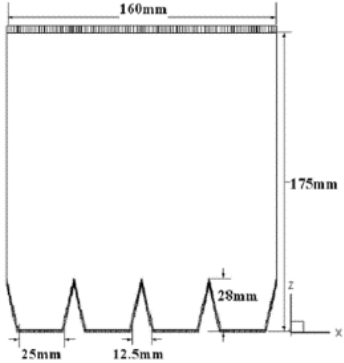
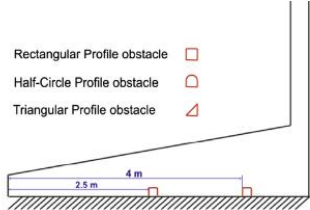
711 Wherein the collector inlet is encircled by a peripheral enclosure, and a series of  
712 axial air fans, controlled by microprocessors, are installed to provide a controllable  
713 flow inlet. Papageorgiou also suggested an electromechanical air stopping system  
714 in the opening to optimize power output by closing low-speed turbines. So that the  
715 residual air could be enhanced by the benefit of its now higher speed. Table 2 shows  
716 the advantages, disadvantages and prominent features of suggested auxiliary  
717 techniques.

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Table 2. Advantages, disadvantages and prominent features of suggested auxiliary techniques

	design	prescription	advantages	disadvantages	improvement	Schema	ref
Flow mixing enhancement	rectangular fin	Effect of longitudinal Continuous and discontinuous type of rectangular fin installation in absorber have been studied	-efficiency improvement by increasing the depth ratio of field (heat transfer area) -heat transfer coefficient improvement due to disturbance and reformation of boundary layer	-maintenance difficulties due to hard access inside the site -excess material usage in fins -higher pressure drop	- thermal performance improvement by the increasing number of fins - efficiency improvement by increasing the depth ratio of fins -performance enhancement in discontinuous type by appropriate interruption gaps		[101]
	Different fin shape	triangular, elliptical and rectangular fins effect in natural convection solar air heaters	-increasing thermal efficiency by redirecting the flow	-manufacturing difficulties in unconventional fin shape -maintenance difficulties due to hard access inside the site -excess material usage in fins -increasing in pressure drop	-The thermal efficiency of the solar air heater with rectangular fins is 5.5% higher than elliptical and triangular fins		[102]
	Passive flow control implementation	different shape obstacle implementation such as half-circle, rectangular and triangular	-altering the flow field and improve mixing inflow by secondary flows and vorticities.	-flow blockage in obstacles	- increasing velocity rate and consequently energy output improvement up to 41.2% - An obstacle with a triangular profile supplies more thermal performance enhancement since the flow		[103]

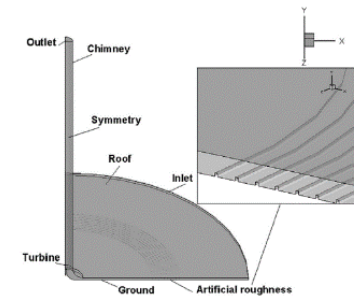
artificial roughness

Considering the optimal location for artificial roughness due to velocity reduction despite heat transfer improvement

-either with natural roughness like ground surface or by artificial roughness such as energy storage system roughness has an inevitable impact  
- it can take advantages of existing component like water pipes  
- the existence of roughness has a positive impact on the performance of the power plant When the wind is blowing

-reduction in velocity

pattern was guided toward the chimney.  
-installation artificial roughness near the collector entrance has a better influence in efficiency



[104]

Incoming radiation increase

Intensifiers utilization

utilizing intensifiers to intensify heat flux and air tank to increase absorption of reflected solar radiation

- higher heat flux entered the solar chimney by utilizing intensifier

-utilization in small SCPPs due to collector shape and intensifier height limitation  
-ray tracing reflectors apparatus difficulties

-two times increment in maximum velocity compared with a without-intensifier prototype



[107]

Reflectors utilization	A SCPP with enhanced incident solar radiation aided with reflectors	-higher input flux temperature and mass flow rate	-limitation in reflectors high in large scale SCPPs	- the gain in efficiency was increased up to 22 percent compared to conventional solar chimney power, ten percent floor temperature increment and 134 percent increment in mass flow rate -133% power output increment
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[109]

Secondary roof supplement

Transformative closure	Transformative end closure in underneath roof
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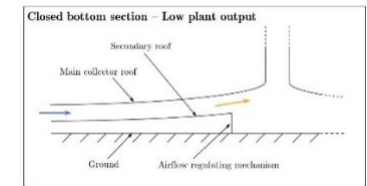
-warm trapped air acting as an energy storage medium

-controllable mechanism difficulty for transformative end closure end

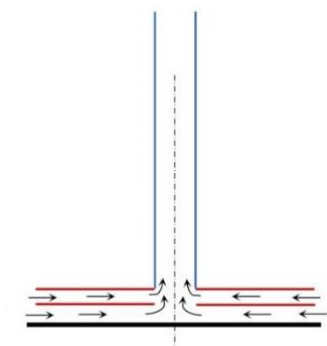
Double roof	Parallel and counter flow mechanism
-------------	-------------------------------------

-more heating due to the longer path in counter flow type and exposed to warm air instead of ambient

-more pressure loss due to longer path



[61]



[105]

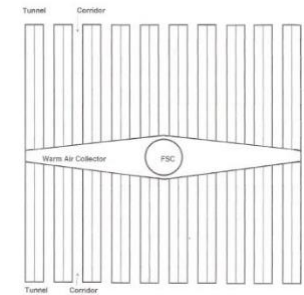
Novel collector design

Modular U type collector

Utilizing a series of long U or V type glasses

-prevent dust layer formation and consequent problems  
-low cost and with lesser working in site difficulties

-higher pressure loss

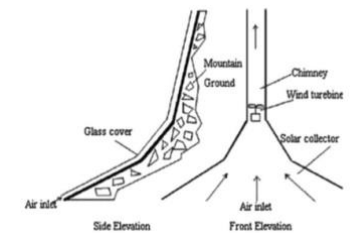


[138]

sloped SCPP

Sloped SCPP along with mountain profile

-better companionship with mountain ups and downs  
-chimney behavior of collector due to rising



[126]

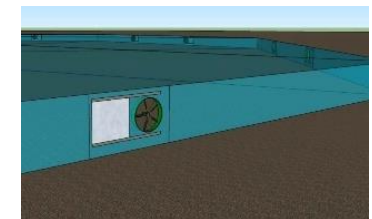
Enclosed SCPPs

A closed collector entering containing a number of the axial fan instead of large scale turbines

-lower cost due to complex turbine section deletion such as gearbox  
-external wind altering  
- optimized power output by a controlled electro-mechanical air stopping system  
- convenient repair and maintenance

-low efficiency fans and numerous number of devices

- crosswind secured thermal storage layer



[137]

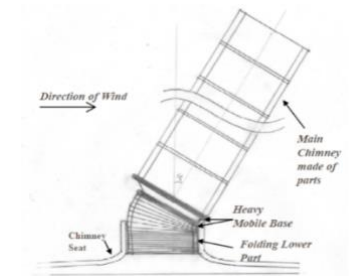
Novel chimney designs

Floating SCPP

Chimney filled with lighter than air gas make the possibility to lift off and tilting in cross wind

-relative negative pressure in outlet due to cross wind and providing good driving force  
-alleviating heavy solid problems  
-alleviating crosswind concerns

-lowering SCPP height in tilting operation mode



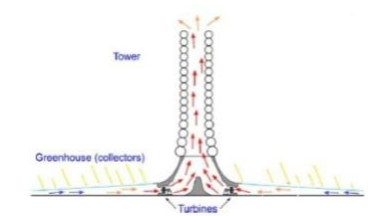
[113]

toroidal bladders chimney

tubes with sliding possibility in cross wind interaction

-relative negative pressure in outlet due to cross wind and providing a good driving force  
-alleviating crosswind concerns

-lowering SCPP height in tilting operation mode



[121]

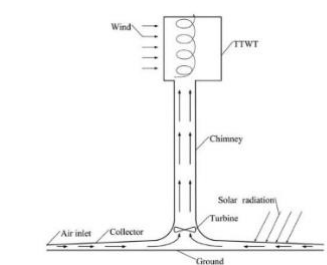
Tornado wind tower combined

Tornado wind tower installed at the outlet to make a favorable driving force

-relative negative pressure in outlet and providing good driving force  
-ability of utilizing smaller SCPPs

-low improvement in low wind speed region

-providing power output up to a twice height size



[122]

765

766 **3.3 Hybrid SCPPs**

767 Because sunlight is not available for all hours of the day, and is also reduced on  
768 cloudy days, scholars have considered how to reduce these effects and allow some  
769 form of 24-hour operation without intermittency. There are reports on combining  
770 technologies, for example, using fuel cells, thermal energy storage, geothermal  
771 effects, photovoltaic cells, wind power technologies etc., with solar, so creating  
772 multi-objective systems such as water desalination and solar drying procedures etc.

773 **3.3.1. Desalination**

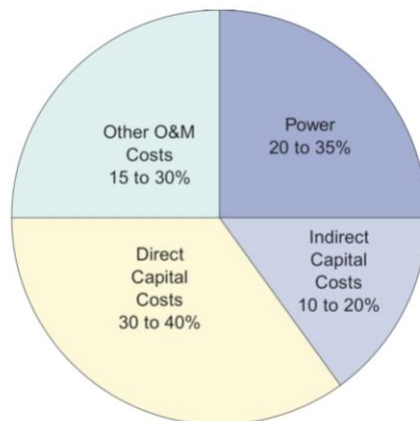
774 Desalination of impure water is one simple, cheap and useful way of using brackish  
775 water. However, water desalination is limited by factors such as the need for large-  
776 area solar distillation plants, low water production rates per unit area and the natural  
777 restriction of limited solar radiation in some regions [139]. According to the report  
778 from International Water Association (IWA) published in 2016, water desalination  
779 can cut costs (as shown in Fig. 23), since it is becoming an effective way of solving  
780 water demand problems in areas with high water salinity level [140-142]. However,  
781 at this time, integrated SCs with water desalination are not yet practical and this  
782 promising hybrid needs further studies. That being said, cost forecasts over the next  
783 15 years, show a considerable reduction in construction costs, electrical energy  
784 usage etc., as shown in table 3.

785

786

Table 3.Costs of desalination for medium and large projects[142].

<b>Parameter for Best-in Class Desalination Plants</b>	<b>Year 2016</b>	<b>Within 5 Years</b>	<b>Within 20 Years</b>
Cost of Water (US\$/m <sup>3</sup> )	0.8 – 1.2	0.6 – 1.0	0.3 – 0.5
Construction Cost (US\$/MLD)	1.2 – 2.2	1.0 – 1.8	0.5 – 0.9
Electrical Energy Use (kWh/m <sup>3</sup> )	3.5 – 4.0	2.8 – 3.2	2.1 – 2.4



787

788

Fig. 23.Desalination cost breakdown[142].

789 Zou et al. evaluated two types of hybrid systems, 1) a wind supercharging SCPP  
790 with seawater desalination and waste heat, 2) SCPP integrated with seawater

791 desalination and waste heat [143]. They investigated the proposed models both  
 792 experimentally and mathematically to estimate the efficiency of hybrid  
 793 systems. The results in the research of Zou et al. are as shown in table 4.

794 Table 4. Comparing results and the evaluated parameters by Zou et al. [143]

<b>Investigated parameter</b>	<b>Results</b>	<b>Comparison</b>
Increase in chimney's height	Water desalination improved Power generation improved	---
Increase in solar irradiance	Water production decreased Generated power improved;	---
Decline in seawater depth while enough solar irradiance exists	Water desalination improved Power generation improved	---
Increase in the temperature of the exhaust gas	Water desalination improved Power generation improved	---
Performance	---	WSCPDPW was better than SCPPDW about 15%

### 795 3.3.2. Drying Technology

796 Solar drying dates back to 8000 BCE, when the first solar dryer was installed in  
 797 France. In an energy conscious world, solar drying is becoming a necessity because  
 798 of its major merits, such as no requirement for fuel or electric power. Drying is an  
 799 activity which consumes lots of energy in its broader productions applications, for  
 800 example, textile manufacture [144], brick production [145], cement production  
 801 [146], and wood and timber treatment [147] etc. Solar drying therefore offers yet  
 802 another option for switching to a more eco-friendly method rather than using fossil  
 803 fuels. Sandali et al. reviewed the enhancement of solar drying system by various  
 804 techniques and factors in multiple solar dryers, such as direct, indirect, mixed-mode  
 805 and hybrid dryers [148]. At the same time, the effect of different climate conditions,  
 806 geometry, heat exchanger and heat pumps, reflector addition, phase change material  
 807 (PCM) and etc. were also evaluated. The results of which, showed that climate  
 808 conditions and solar radiation had the greatest impact.

809 Afriyie et al. investigated the performance of a solar dryer experimentally [149]. In  
 810 their research they performed their tests firstly in a cabinet dryer, followed by tests  
 811 with a chimney. Eventually, the trials were conducted with a tent dryer in which the  
 812 roof of the drying chamber was inclined. Afriyie et al. also monitored effective  
 813 factors such as air velocity, temperature, relative humidity and the moisture content  
 814 in a crop.



815 Afriyie et al. divided their experimental tests into two different categories, namely  
 816 no-load and under-load tests. In the under-load category, there were 4 different  
 817 subcategories as follows:

- 818 • Test-set 1: tests on the dryer with roof angle 81°, using the normal chimney.
- 819 • Test-set 2: a repeat of Test-set 1, but with the solar chimney.
- 820 • Test-set 3: using the roof angle of 64°, still with the solar chimney.
- 821 • Test-set 4: using the roof angle 51°, still with the solar chimney.

822 Under-load tests were conducted in the presence of the root crop cassava.

823 In table 5 shows the dryer no-load conditions.

824

825 Table 5. Conditions of the dryer in no-load tests [149].

	No absorber		With chimney absorber	
	Test-set 1	Test-set 2	Test-set 3	Test-set 4
Roof angle (°)	81	81	64	51
Ambient air temperature (°C)	21.83	23	22.17	23.5
Dryer exit air temperature (°C)	26.83	31.33	29.5	30.83
Inlet air relative humidity (%)	57.17	42.17	51.67	42.33
Dryer exit air relative humidity (%)	38.17	26	34.17	24.83
Ambient air velocity ( <i>m/s</i> )	0.02	0.01	0.02	0.01
Dryer inlet air velocity ( <i>m/s</i> )	0.14	0.18	0.19	0.2
Dryer exit air velocity ( <i>m/s</i> )	0.39	0.45	0.49	0.52

826 Afriyie et al. concluded that a solar chimney crop dryer is advantageous because it  
 827 is cheap to construct, and that its performance is enhanced when the relative  
 828 humidity of the ambient air falls. However, conversely, performance is reduced  
 829 when relative humidity is high [149].

### 830 3.3.3. Photovoltaic Cells and SCPPs

831 Rahbar and Riasi proposed novel configurations of conventional solar chimney  
 832 power plants (CSCP) coupled to a photovoltaic cell (PVSCP) and brackish water  
 833 desalination (PVDSCP) in order to utilize solar energy more effectively [150]. In  
 834 their study, Rahbar and Riasi developed a 1-D mathematical model for each  
 835 configuration and then validated their mathematical results by experimental results.  
 836 The studied geometry is shown in the Fig. 24.

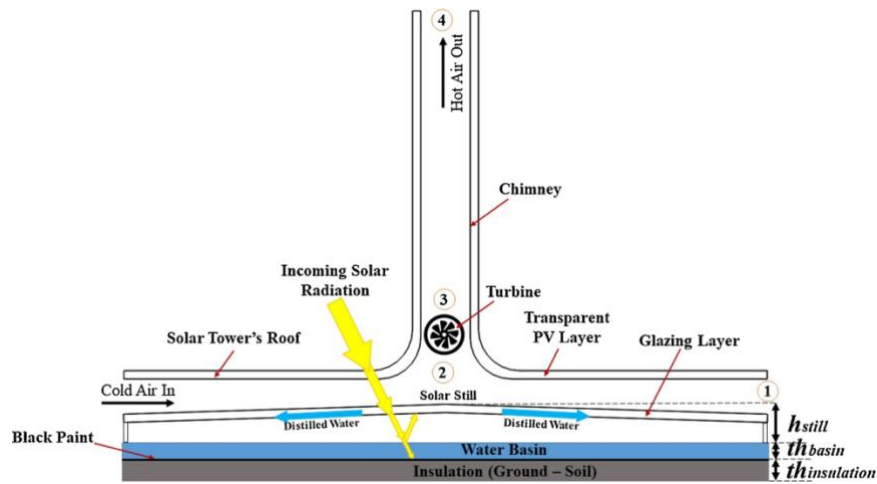


Fig. 24. Schematic detailed geometry of the PVSCP and the PVDSCP [150].

837  
838

839 From their results, it is revealed that a PVDSCP is more efficient than a CSCP and  
840 a PVSCP by 26.13% and 21.92%, respectively. In contrast, the efficiency of the  
841 turbine is higher than the CSCP and PVDSCP by about 17.9% and 31.3%,  
842 respectively. Key parameters in their optimization were the roof radiation, roof  
843 height, tower radiation, tower height, and mass flow rate. After comparing the  
844 optimization with the Manzanares solar chimney power plant[36], their results  
845 showed that utilizing a PVSCP and PVDSCP in Manzanares could improve the  
846 efficiency of the power plant, 55.97%, and 71.8%, respectively.

847 Using fluid dynamic analysis Haghight et al., evaluated different PV panels in four  
848 different locations in a SC [151], and their investigated parameters (as shown in Fig.  
849 25), included the location and the widths of the PV panels within the SC. Three  
850 different PV widths of 70, 50, and 30 cm were tested and the 50 cm cell proved to  
851 offer the best results, in that when the transparent collector was replaced with the  
852 50cm width PV cell, the efficiency is increased by 1%.

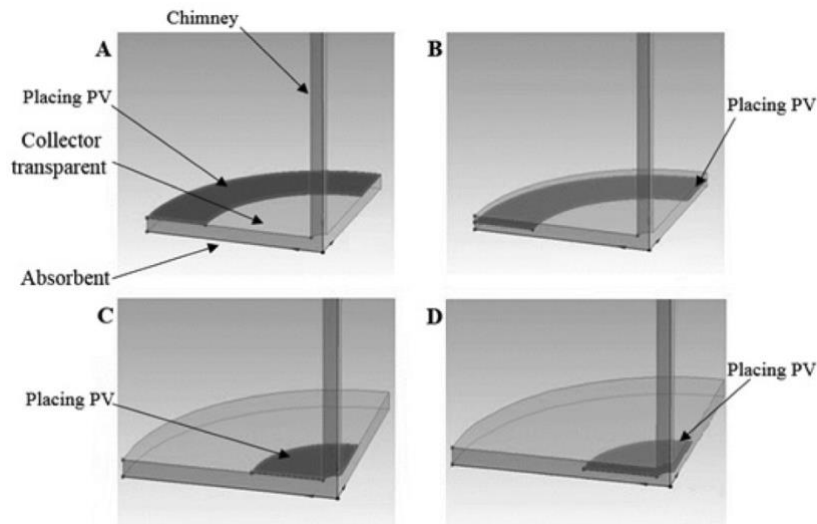


Fig. 25. Investigated parameters by Haghight et al. [151].

853  
854

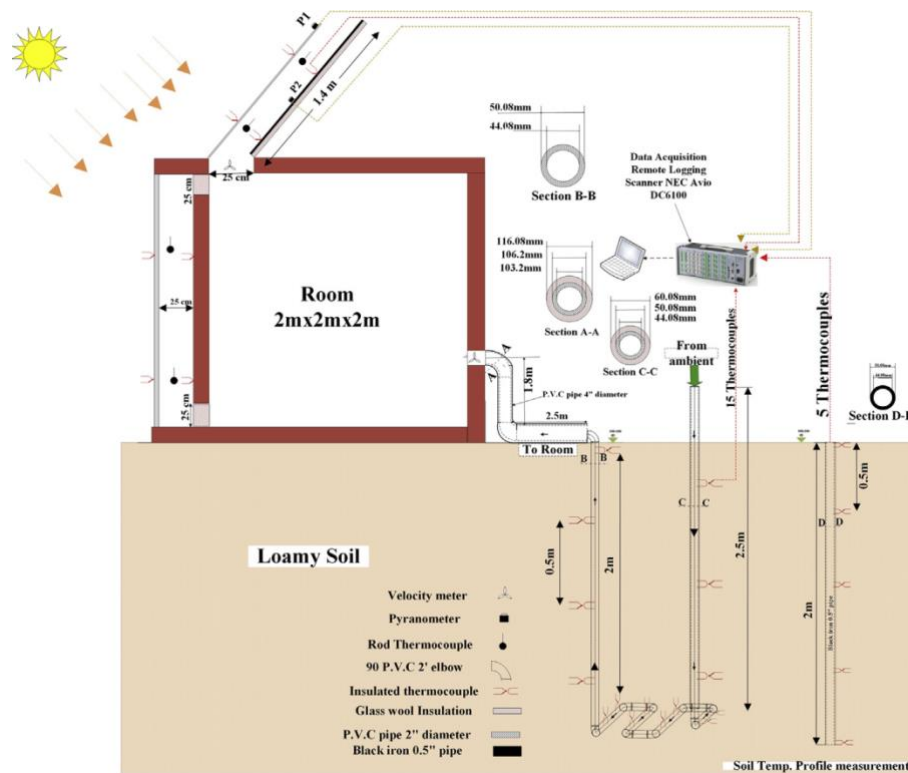
855 Ahmed et al. proposed a new design for a chimney in Kirkuk, Iraq [152] by  
856 constructing two types of experimental model. In the first model, the collector roof

857 is made of a glass and a PV panel was installed as an absorber, whilst in the second  
 858 model, the collector roof was made of PV panels and used plywood as the absorber.  
 859 They observed that the useful power from the second model's was greater.

### 860 3.3.4. Ventilation

861 In applications related to ventilation, the most examined parameter is the air flow  
 862 rate. Nguyen and Wells analyzed the flow rate and thermal efficiency of using  
 863 hybrid SCs to ventilate buildings through CFD simulation [153], their evaluation  
 864 dimensions being chimney length, air channel gap, inlet height, outlet height, inlet  
 865 width, and outlet width. Results showed an increase in flow rate is possible with  
 866 increased absorber surface length, air channel gap, and the heat flux, but a drop in  
 867 thermal efficiency for the outlet width – due to flow reversal at the outlet.

868 Serageldin et al. performed a parametric study, which included optimization  
 869 methods for heating and ventilation systems through CFD, as shown in Fig. 26  
 870 [154]. In this research, they validated their results against an experimental result in  
 871 the cold season on March 14-22, 2016 in Egypt.



872 Fig. 26. Schematic of the studied geometry by Serageldin et al. [154].  
 873

874 In their investigation, they optimized the system through the central composite  
 875 design of the experiment (CCD) algorithm, and found eight parameters to maximize  
 876 the ventilation rate for the solar chimney configuration and earth-to-air heat  
 877 exchanger design, these being: width, length, air gap, inclination angle, position and  
 878 pipe diameter, inlet position, and inlet height. The most sensitive parameters being  
 879 (in order), EAHE pipe diameter, chimney height, EAHE height and position, solar  
 880 inclination angle, width, and gap. They also concluded that the optimum chimney

881 inclination angle, length, width, and gap, lie within ranges of 30–35°, 1.94–1.97 m,  
882 0.92–0.97 m, and 0.19–0.23 m, respectively.

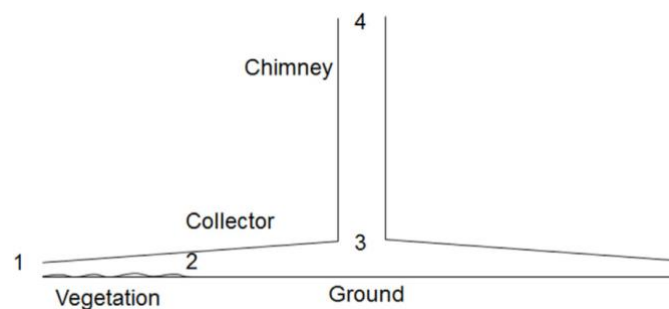
883 Kong et al. examined the variation of the inclination angle of a SC installed on a  
884 roof, looking to enhance its performance for ventilation purposes using CFD based  
885 methods. The work considered two cities in Australia – Adelaide airport, Darwin,  
886 and Townsville Aero [155], and Kong et al. performed their simulations under four  
887 different heat fluxes, 200, 400, 600 and 800 W/m<sup>2</sup>, together with the impact of  
888 various inclination angles, 30°, 45°, 60°, 75° and 90°, respectively. They reported  
889 two main conclusions as follows: 1) In real-life related applications the inclination  
890 angle exerts a profound influence on the received solar irradiance and the ventilation  
891 efficiency. 2) In their numerical evaluation, the greater inclination angle, the greater  
892 the space ventilation.

893 Wang et al. described a hybrid SC for ventilation [156]. They examined 4 main  
894 parameters through a CFD approach, namely the thickness of the glass panel, the  
895 width of the air gap, the thickness of the water columns and surface tinting. Their  
896 results showed that by cutting the thickness of the glass panel in half, the ventilation  
897 was enhanced by 7.3%. In addition, that up to 0.2 m increase, either in the width of  
898 the air gap or the thickness of the water column, boosted the ventilation rate by 21%.

### 899 3.3.5. Power Generation

900 Electrical power is generated through a complex heat transfer process where solar  
901 energy is converted to electrical power. Coupling turbine blades with a SC can  
902 generate power which is a free and durable source for power generation and  
903 Tingzhen et al. numerically analyzed a SC coupled with a turbine [157]. In their  
904 study, they investigated the effect of turbine rotational speed on the outlet of the  
905 chimney. They also considered, by simulation, the design and performance of a  
906 MW-graded SCPP with a 5-bladed turbine. The results showed an efficiency of 50%  
907 at a power output of 10 MW.

908 Xu and Zhou developed a mathematical model for their performance investigation  
909 into a modified solar chimney power plant (MSCPP) used for power generation and  
910 vegetation growth purposes, the schematic for which is shown in the Fig. 27 [17].  
911 The evaluated parameters were solar radiation, ambient temperature, relative  
912 humidity, and chimney height.



913  
914  
915

Fig. 27. Schematic of modified solar chimney power plant, vegetation is grown in some parts of the collector [17].

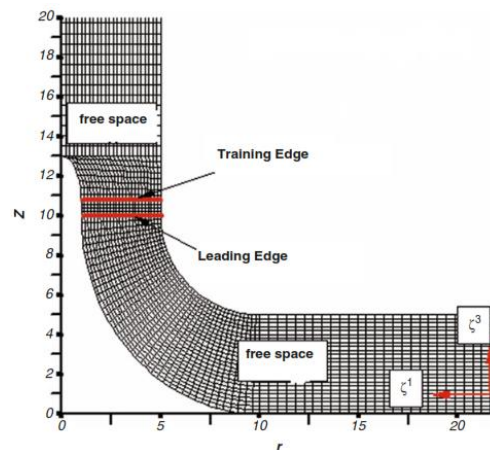
916 They validated their simulation results via the experimental results of Haaf [36].  
917 Results show that an increase in vegetation area results in an increase in the mass  
918 flow rate of the vapour and leads to a considerable reduction of the power.  
919 Furthermore, when the weather is cooler, the production of power also falls. Further,  
920 when the relative humidity is higher, the mass flow rate of the evaporated vapour  
921 from the vegetation area decreases, but power generation increases. Finally, they  
922 concluded that a modified solar chimney power plant is more advantageous than a  
923 conventional SCPP.

924 Wahab and Al-Maliki studied the capability of utilizing a SCPP in a farm to produce  
925 electrical power for agricultural demands, such as pumping systems, irrigation, and  
926 lighting etc. The study was performed using Ansys software and was compared with  
927 the results obtained from an experimental farm condition [158]. Wahab and Al-  
928 Maliki concluded that the experimental farm SCPP, with collector dimensions of 70  
929 m  $\times$  50 m and the chimney height of 20 m, had produced electrical power at its  
930 highest level in a year, at about 29 kW [158].

931 Tian et al. analyzed a hybrid SC in a desert city, Yazd, located in Iran [159]. In this  
932 research, they optimized the SC economically using a deer hunting optimization  
933 algorithm whilst comparing these optimization results with the genetic algorithm  
934 and particle swarm algorithm of MATLAB software. Tian et al. proposed two days,  
935 in July and in February, in order to analyze the hottest and the coldest days in the  
936 desert region of Yazd. From the results, they concluded that on the hottest day in  
937 July, owing to the high intensity of the sunlight, the produced energy is comparable  
938 with the coldest day in February.

### 939 3.3.6. Wind Turbine

940 Negrou et al. investigated the thickness distribution of blades and the desired swirl  
941 distribution [160] which was proposed by Wu [161]. The goal of their research was  
942 to design the turbine blades, based on the dimensions of the existing prototype  
943 suggested by Manzanares [5], which then speeds up the optimization design for a  
944 SC. Also, incompressible non-viscous flow was studied as part of this their research.  
945 Fig. 28 shows the studied geometry and the generated mesh in their model.



946  
947 Fig. 28. Studied geometry by Negrou et al. [160].

948 **3.3.7. Cooling Systems**

949 Nasri et al. designed a new system based on the integration of a SC and a solar-air  
950 conditioning system [162]. The system works upon the principles of adsorption  
951 chilling and desiccant dehumidification, and as part of the investigation, the authors  
952 theoretically considered the proposed system under real conditions related to  
953 Tunisia. The test was performed on 4 different days, and on each occasion the air  
954 temperature decreased, and the relative humidity increased during the pre-cooling  
955 and cooling processes, respectively.

956 Hweij et al. studies the efficiency enhancement of cooling a window using a solar  
957 chimney, analyzing and predicting the window temperature and its impact on  
958 comfort under the conditions in an office space located in Riyadh, Saudi  
959 Arabia[163]. There were two representative hours, 14 h and 17 h, for this hot, dry  
960 climate. The results showed that thermal comfort was improved by the provided  
961 system. When the system was present at 14 h, the thermal comfort reaches 1.42.  
962 However, and for comparison when the proposed system was absent, in order to  
963 reach the same level of thermal comfort (1.42), a similar space was considered  
964 where all the conditions were constant, but the supply temperature was kept  
965 variable. Likewise, when the system was present at 17 h, the comfort level was 1.96  
966 The work showed that the proposed system saves energy by approximately 10%  
967 (see Table 6).

968 Table 6. Saved energy at 14 h and 17 h [163].

Cases	Overall thermal comfort	Energy-saving (%)
14 h	1.42	9.64
17 h	1.96	9.8

969

970 **3.3.8. Water Harvesting**

971 Ming et al. suggested a modified solar chimney that can generate freshwater in  
972 addition to generating power [164] and the proposed chimney design has undergone  
973 experimental tests within nine different cities in China. The work looked at the moist  
974 air which condenses above the lifting condensation level, and of a one-dimensional  
975 compressible flow and the heat transfer mathematical model developed. The work  
976 showed a direct correlation between the precipitation of the environment and the  
977 produced water by this system – in general the modified system increasing water  
978 production by a coefficient of 0.875. Furthermore, they observed that the system is  
979 also effective in arid regions, and Table 7, indicates a convincing increase in the  
980 ratio of water production to natural precipitation across all nine cities.

981

982

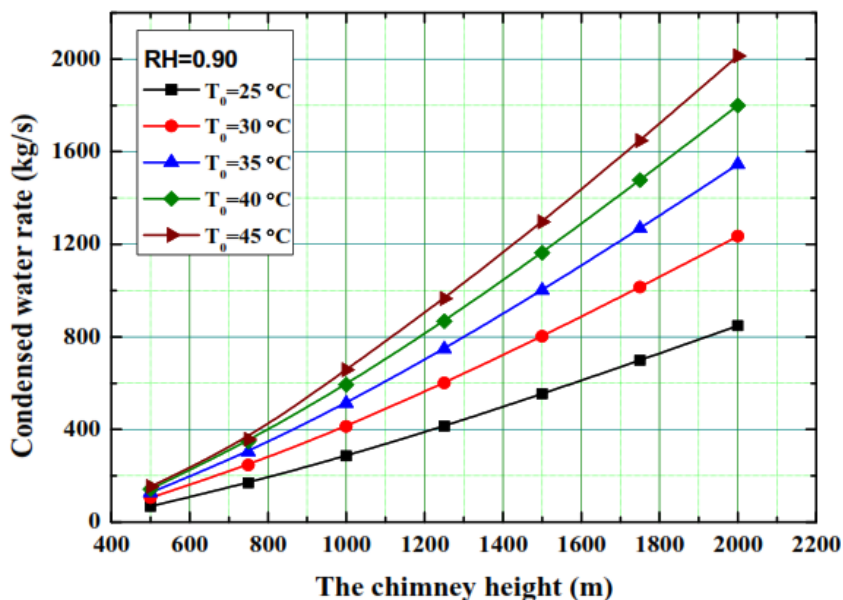
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985

Table 7. Natural precipitation (NP), water production (WP), and the ratio of WP/NP at nine stations in 2013 [164].

	NP (mm)	WP (10 <sup>4</sup> t)	WP (mm)	Ratio (WP/NP)	Sunshine duration (h)
Chengdu	1343.3	29.71	5942	4.42	1128.8
Shanghai	1173.4	23.62	4724	4.03	1864.7
Shijiazhuang	508.3	17.08	3416	6.72	1716.8
Zhengzhou	353.2	12.87	2574	7.29	1925.6
Wuhan	1434.2	29.37	5874	4.10	2092.5
Chongqing	1026.9	27.40	5480	5.34	1213.7
Beijing	579.1	14.59	2918	5.04	2371.1
Urumqi	300.9	8.18	1636	5.44	3068.6
Guangzhou	2095.4	37.92	7584	3.62	1582.9

986 Wu et al. proposed a modified solar chimney power plant, called a “Aero logical  
987 Accelerator” (AeAc), which utilizes the latent heat of a condensation process [165].  
988 Using a mathematical model, the potential energy and generated water at different  
989 ambient temperatures were conducted, and their results (Fig. 29), show that the  
990 system can generate both water and electrical energy for domestic usage if the right  
991 method is used for collecting the water. As such, electrical energy generation  
992 depends on the temperature of the entering heated air and the system size.  
993 Furthermore, water generation is determined by the relative humidity at the entrance  
994 of the chimney in addition to the aforementioned factors.



995  
996

Fig. 29. Rate of condensed water vs. chimney height at RH= 0.90 [165].

997 Hoseini and Mehdipour numerically analyzed the effect of solar radiation and water  
998 temperature on power generation and water harvesting for two different cases,  
999 namely (1) integrating a SC with a humidifier and (2) coupling a chimney with a

1000 humidifier and a condenser [166]. In the conducted research, they concluded that,  
1001 in the first case, by utilizing a hybrid SC, the generated power rises with increasing  
1002 solar radiation, and that it increases at least 1.3 times higher than with a typical SC  
1003 when the water temperature is at a minimum of 10 °C. With the second case, they  
1004 found that as solar radiation grows, the amount of harvested water declines.

1005

#### 1006 4. Conclusion

1007 Solar chimney power technology is a simple thermal technology involving three  
1008 main components: The Solar Chimney, the Solar Collector and a Power Conversion  
1009 Unit. However, despite being a straightforward and eco-friendly technology, its low  
1010 efficiency remains an important practical drawback. Accordingly, solutions to this  
1011 problem have drawn much attention in order to harvest greater power output and  
1012 lower spending costs. This work aims to present a comprehensive, and for the most  
1013 recent, detailed and applicable studies including experimental, theoretical,  
1014 simulation and reviews which cover new design concepts along with configuration  
1015 suggestions for efficiency increments.

1016 The crucial point that emerged from the experimental studies, is that most of the  
1017 constructed SCPPs are built at a small scale, and are not capable of utilizing  
1018 potential improvements that are inherent with larger economies of scale because  
1019 most of the system parameters are strongly dependent on the constructional  
1020 problems that would occur with larger scale plant, such as chimney divergence  
1021 angle. However, despite their low efficiency (for SCPPs, less than 2%), some  
1022 researchers have focused on improving the overall efficiency through the  
1023 enhancement of individual parts of the system. Hence with such an aim in mind,  
1024 many design parameters have been considered within each section, including  
1025 changes to profile – both chimney and collector segment. Novel ideas such as  
1026 utilizing fins, obstacles, reflectors and a secondary roof for the collector, for a  
1027 floating chimney, for different profile shapes and new chimney configurations have  
1028 been reviewed as well.

1029 In some studies, the low-efficiency imperfections have been reduced by integrating  
1030 an SCPP other systems, and these are categorized into 8 main groups, namely: 1)  
1031 Water Desalination Systems, 2) Drying Products, 3) PV Collectors, 4) Ventilation  
1032 Systems, 5) Power Generation Systems, 6) Geothermal SC, 7) Cooling Systems,  
1033 and 8) Water Harvesting. Water desalination can reduce the costs of providing water  
1034 in some arid regions. The integration of SCPPs and dryers is **not** an option since  
1035 solar energy is an eco-friendly, durable and free source. The other types of hybrid  
1036 SCPPs are power generating SCPPs which can generate electricity from the sun by  
1037 a simple replacement in the collector, e.g. installing turbine blades in the SC.  
1038 Ventilation in buildings and chilling systems is another usage of hybrid SCs which  
1039 can enhance the efficiency of the ventilation and the cooling system in addition to  
1040 generating free power. The other important function of hybrid SCs is water  
1041 harvesting that is one possible solution to the ever-increasing problem of the water  
1042 crisis which is spreading across the world.

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