

On the role of Poisson's contraction propagation in the creation of squeeze-film assisted levitation

Advances in Mechanical Engineering
2023, Vol. 15(10) 1–8
© The Author(s) 2023
DOI: 10.1177/16878132231194277
journals.sagepub.com/home/ade



Cristinel Mares¹, Masaaki Miyatake² and Tadeusz Stolarski¹ 

Abstract

The paper presents results of computer simulation and experimental observations with the main objective of explaining the role of Poisson's contraction phenomenon in the mechanism of initiating and maintaining squeeze-film action through which light and planar objects can be levitated. Initial and centrally located shallow dimple, created in a thin plate under plain tensile stress is, with the passage of time, transformed into a pattern of elastic deformations over plate's surface. As a result of that, a light object levitates not only at the centre of the plate where initial, primary dimple is located but over the entire plate.

Keywords

Poisson's contraction, squeeze-film, levitation, elastic wave propagation

Date received: 2 March 2023; accepted: 25 July 2023

Handling Editor: Chenhui Liang

Introduction

Conventional contact methods of manufacturing conveying are quite unsuitable for light and fragile objects and additionally inappropriate for special stringent operating conditions, such as for food, drug and semiconductor chip manufacture. Problems with contact conveying also include conveying rough materials that can reduce the life of the conveyor. Another challenge for contact conveying is sticky materials, which tend to stay on the conveyor beyond the discharge point and requires suitable cleaning and/or the conveyor to have additional mechanisms for removing the sticky material. Corrosive items should be conveyed on special corrosion-resistant belts and elaborate belt cleaning design may be necessary. Any additional mechanisms added to deal with the problems described above will add cost to a contact conveying system.^{1–3} Wear can occur at conveyor loading points, which creates trace quantities of substances like metal debris from abrasion. When conveying high-purity materials or delicate objects such as pharmaceuticals or electronic parts the

acceptable levels of contamination are very low. Thus, the conveying of such material in industry requires rigorous cleaning and sterilisation processes. For example, in semiconductor technologies, it is challenging to convey a component of a Micro-Electromechanical System (MEMS) due to their delicate surface characteristics and fragility. Conventional assembly procedures are typically based on mechanical contact, which can lead to damage of fragile components or some degree of surface damage. Therefore, these circumstances would benefit from handling such fragile components without physical contact.^{4,5}

¹Department of Mechanical and Aerospace Engineering, Brunel University London, Uxbridge, UK

²Department of Mechanical Engineering, Tokyo University of Science, Tokyo, Japan

Corresponding author:

Tadeusz Stolarski, Department of Mechanical and Aerospace Engineering, Brunel University London, Uxbridge, Middlesex UB8 3PH, UK.
Email: mesttas@brunel.ac.uk



For the handling or conveying of delicate and light-weight objects, for example, electronic chip parts or silicon wafers during manufacture, the microelectronics industry utilises robotic arms, conveyers and chucks, which cause mechanical attrition with the substrates. Such interaction produces particles, polluting the highly controlled work location and thus, significantly affect the product. In other industrial applications such as food packaging, sorting of postal items and pharmaceutical products, objects on a conventional production line are moved by a flexible belt, manipulator or other mechanical means such as vibratory conveyors, all of which involve contact forces with the risk of damage or friction losses. Delicate food products such as biscuits or aluminium beverage cans are currently conveyed by vibratory conveyors and air beds with consequent damage plus the production of dust, which causes problems. Levitation is the principle whereby the lifting force is applied to the conveyed item without physical contact. Five distinct techniques of levitation are identified in the literature, namely, magnetic, electric, optical, aerodynamic and acoustic.

To deal with these problems, the squeeze-film levitation (SFL) technique has been proposed, thus permitting controlled lifting and transportation of objects without any mechanical contact.

In the field of micro-assembly, food, drug, accurate measurement and semiconductor chip fabrication, precision positioning and carrying delicate items is essential. The classical conveying process, which is generally based on mechanical contact, may contaminate and damage transporting objects. Traditional non-contact conveying technologies, like magnetic systems and air cushions, also creates various drawbacks in their applications such as particle accumulation and provide of a large volume of clean air necessary for air cushions considerably rises the cost of their operation⁵⁻⁷). Squeeze Film Levitation (SFL) is anticipated to be an attractive contactless conveying technology since it has many advantages, for example, any material conductor or insulator, magnetic or non-magnetic, can be handled.^{8,9} The design of SFL can be as compact using a set of piezoelectric actuators attached to the underside a driving surface (thin plate). These distinct features make it appropriate for particular conditions where the compressed air or magnet sources for levitation and/or conveying are not applicable.

Squeeze-film technique and resulting levitation was first presented by Langlois¹⁰ in 1962 and Salbu¹¹ in 1964. The essence of it is that a time-average pressure in a gap between interacting surfaces is greater than the ambient pressure, caused by the second-order influences resulting from a quick compression and de-compression of a gas film existing between the surfaces. Theoretical analysis of the squeeze-film action is based on the Reynolds equation. In 1975, Whymark¹² described a

squeeze-film levitation design comprising of a piston vibration at an ultrasonic frequency of 20 kHz in order to levitate a flat brass disk with diameter of 50 mm. Wiesendanger¹³ proposed a simple model that describes the basic idea of squeeze-film levitation performance. Only an entrapped gas between two surfaces rapidly compressed and decompressed was considered, and leakages at the domain boundary were neglected. The harmonic motion of the driving surface results in a non-harmonic pressure variation within the gas film which average value is greater than the ambient pressure.

In general, the squeeze-film levitation method utilises the Bernoulli principle to generate a positive pressure within a film of air existing between a base surface and levitating object (e.g. an electronic chip or silicon wafer). This is, essentially, about forcing a flow of fluid (squeezing) from the gap existing between two surfaces, that is the base surface and levitated object. That can be accomplished in a number of different ways. A classic example of squeeze-film action is described by the Reynolds equation, which envisages the case when one of the surfaces is stationary and the other one moves toward it at the right angle with an appropriate velocity thus giving rise to a pressure within the separating gap sufficient to support conveyed object.¹⁴

As the fluid mechanics fundamentals of squeeze-film phenomenon are well established and can be found in a number of publications,^{15,16} therefore the emphasis of this paper is, instead, on the mechanism initiating and maintaining squeeze-film action. One of them is the use of Poisson's contraction, a well-known physical phenomenon,¹⁷ to create a shallow dimple in a thin and flat plate with the geometry satisfying the basic requirement of squeeze-film mechanism.

The essence of the mechanism consists in the fact that a thin plate subjected to a uniaxial load creating plane stress will deform in the direction perpendicular to the applied load. This is well known and widely accepted Poisson's contraction phenomenon.

The size of the dimple and its depth are both controlled by the magnitude of applied load, geometry of the plate and mechanical properties (Young's modulus and Poisson's ratio) of the plate's material. The shape of the dimple is such that it conforms to the requirement of the squeeze-film mechanism, that is when a planar and light object is placed on the plate at the location of the dimple the gap converging in the direction of fluid (air) flow is created.

To induce squeeze-film action the dimple must be cyclically formed. This can be achieved using piezoelectric transducers, usually called PZT actuators. Oscillation of the dimple, created by Poisson's effect, is forcing the fluid (air) entrapped within the gap existing between the plate and a flat floating object to flow outside and inside in accordance with cyclic deformation of the plate. However, under steady-state conditions

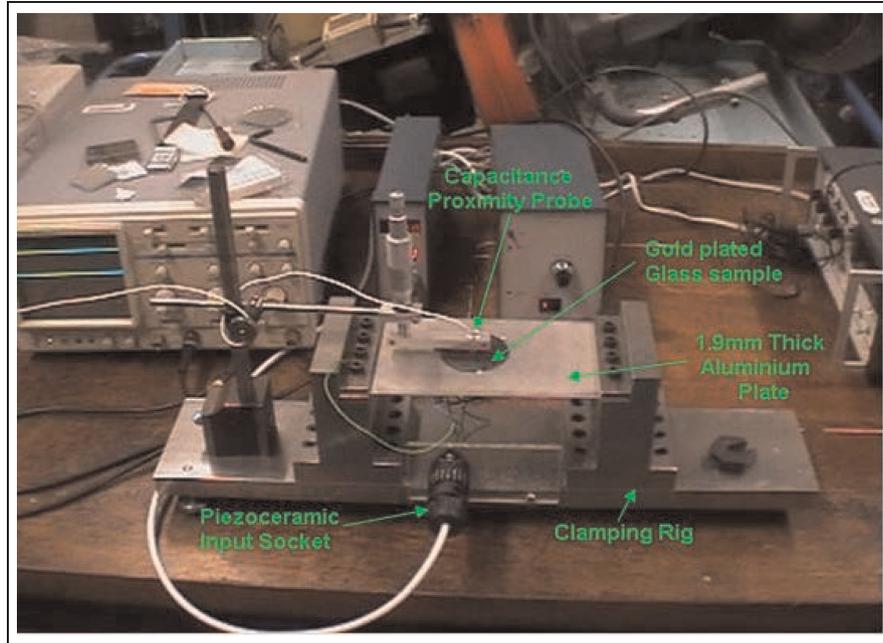


Figure 1. Photograph of the experimental test rig.

established between pulsating dimple and the planar object, the average pressure within the air film is greater than the ambient pressure.

This simplified description of the squeeze-film action leading to floating of a light, planar object suggests that only over the centrally positioned dimple the levitation can take place and nowhere else. However, experimental observations suggest that something more complex is happening as the planar object levitates above the plate not only at the location of the central dimple but also all over the plate's surface, although slightly less effectively. Therefore, a search for explanation accounting for experimentally observed facts was undertaken and its findings are presented below.

Experimental background

Main components of experimental set-up

Experimental set-up was designed and assembled for the specific aim to investigate the role of Poisson's contraction in levitation mechanism of light objects.

The test rig is made of a mild steel and holds the thin plate clamped at both ends to a supporting structure. The plates used in experiments were made of aluminium with 1 and 1.9 mm thickness. The plates needed to be flat and smooth.

Figure 1 is a photograph of the experimental arrangements. The main components of the set-up are clearly visible and provide enough information concerning hardware used to investigate levitation due to

Poisson's contraction effect of a thin plate rigidly fixed at both lengthwise ends.

Electronic equipment of the rig, shown schematically in Figure 2, consisted of an 110 V transformer and a sine wave signal generator, operating in the frequency range 10–60 kHz. The 110 V transformer was required to drive the ultrasonic PZT amplifier and the PZT monitor.

The main objective of testing was to ascertain the height of levitation of a light object for a given frequency of dimple oscillation and the voltage of the current powering PZT. A high-fidelity contactless capacitance sensor was used to determine the actual levitation height for given parameters of an experiment.

Testing procedure

Before a test, the plate was thoroughly cleaned with a laboratory solvent. Afterward, the plate was firmly constrained in the test rig base. Next, the levitation object, in the form of a gold-plated glass sample (see Figure 1), was placed on the plate just above the location of PZTs. Dedicated power supply driving the PZT is set to required voltage that is V_{off} and V_{amp} . Signal generator is then adjusted to produce frequency of dimple oscillation set for a given experiment. Afterward, the non-contact distance probe is positioned above the levitating object to measure its floating height. However, as the floating object was moving in an uncontrolled way all over the plate, reason for that is given later, it was necessary to constrain its movement in order to keep it in the proximity of the contactless probe.

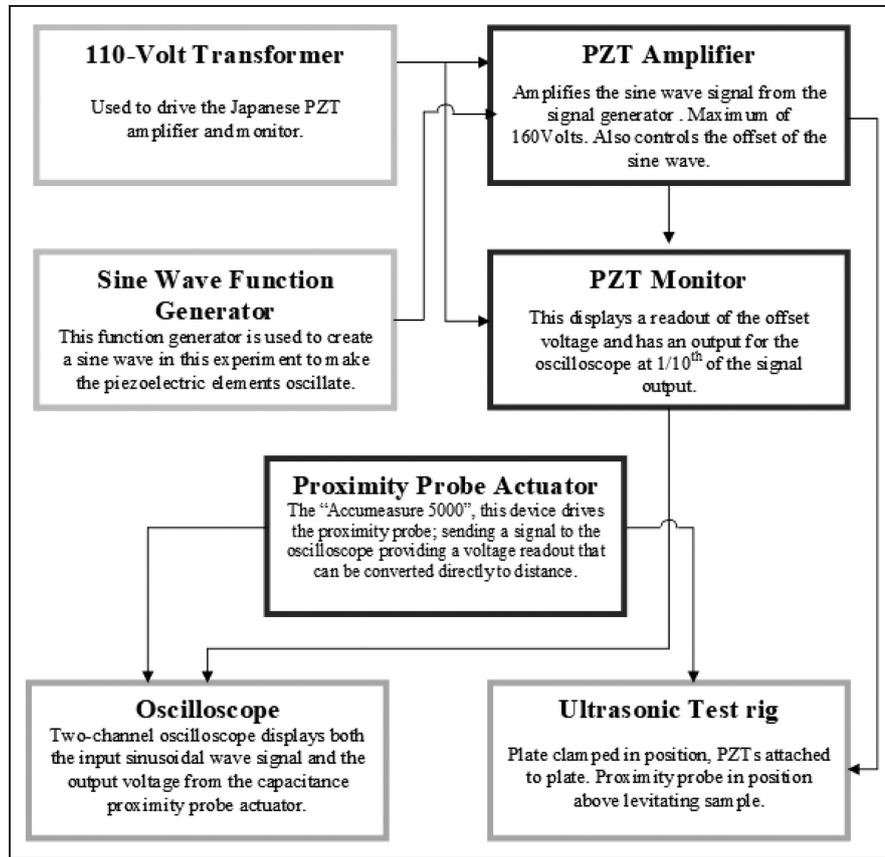


Figure 2. Schematic showing electronic instruments attached to the rig.

Summary of experimental results

Figure 3 shows the most important experimental results where levitation height of the floating object is a function of the object's mass.¹⁸ As expected, the levitation height is decreasing with the increase in the object's mass. This figure gives an idea of the magnitude of load supporting capacity which the Poisson's contraction effect is capable of creating. It also informs of the practical application potential of a device operating on the principle of squeeze-film mechanism generated by the Poisson's phenomenon.

Also, during experimental testing, it was noted that the light object can levitate not only above the location of PZTs where initial dimple due to Poisson's contraction is created but also nearly over the entire surface of the plate. To investigate this somewhat puzzling observation, the plate was covered with a fine powder which particles did not coagulate due to electrostatic attraction. Figures 4 shows experimentally observed pattern formed by the particles on the plate's surface at frequency of PZTs oscillation equal to 7446 Hz.

Figure 5 is an image created by harmonic analysis of the plate under the action of PZTs at the frequency of 7437 Hz and the cyclic load with minimum of 5 N and

maximum of 15 N. Similarity between the experimental observation and computer simulation is unmistakable and supports the argument developed later that cyclically repeated Poisson's contraction is fuelling the spread of elastic deformations over the surface of the plate. The ridges and vales seen in the above images correspond to a pattern of deformations created on the plate's surface.

Results of computer simulation

Setup of simulation

The plate was made of aluminium and had dimensions $140 \times 70 \times 1$ mm. Transient analysis was carried out using ANSYS. The plate was modelled as a 3D solid object made of aluminium with $E = 68$ GPa, Poisson's ratio coefficient $\nu = 0.33$, and density of 2600 kg/m^3 . Solid 185 elements were used, and size of the element was set to 1 mm. All degrees of freedom were applied to the shorter ends of the plate. Forces of cyclic nature and generated by the PZTs were applied to nodes as shown in Figure 9. The equation describing the change of force magnitude with time had the following form as required by ANSYS parameters.

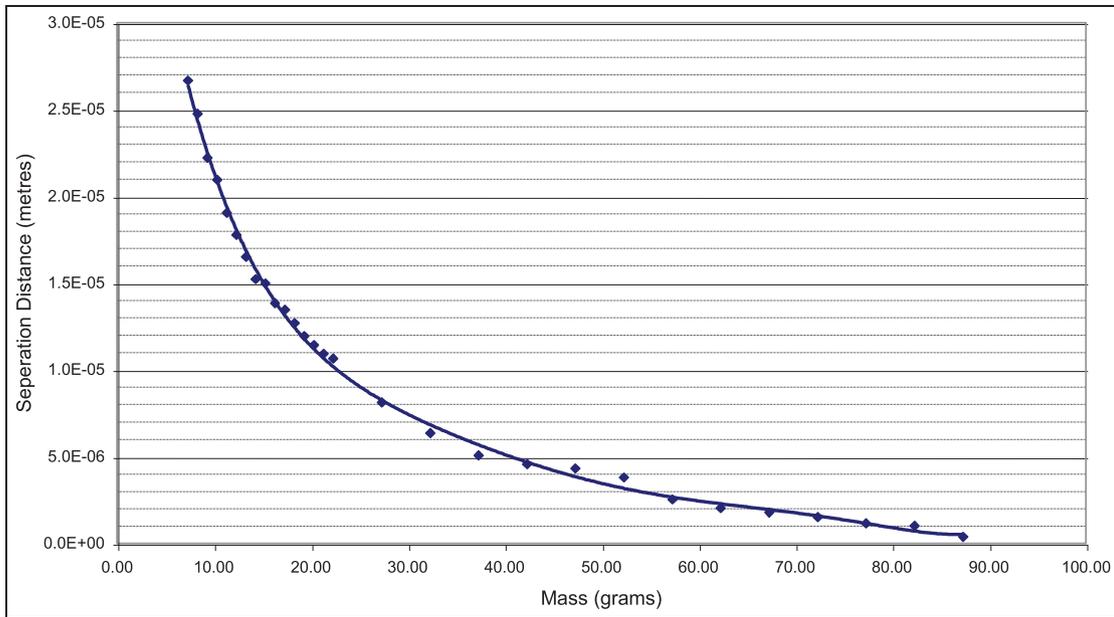


Figure 3. Experimentally determined load supporting capacity.

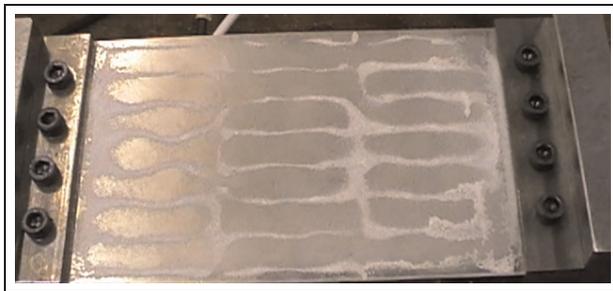


Figure 4. Photograph of the pattern formed by small particles on the surface of the plate.

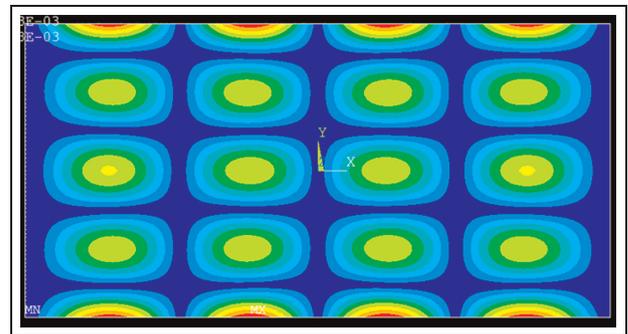


Figure 5. Deformation patterns of the plate resulting from harmonic analysis.

$$Y = 5 * \sin\left(\frac{\pi}{2} * \{TIME\}\right) \pm 10$$

The equation, when plotted using graph plotting facility available in ANSYS, produced cyclic load as shown in Figure 6.

As can be gleaned from Figure 6, the mean load was equal to ± 10 N and the amplitude load was ± 5 N, resulting in the maximum load of 15 N and the minimum load of 5 N. Vertical axis denotes the load in [N] and horizontal axis indicates the number of load cycles shown in the graph. Figure 6(a) shows load applied to the positive X (to the right) and Y (upward) directions, while Figure 6(b) depicts load applied to negative X (to the left) and Y (downward) directions. As a result of that the plate was subjected to a uniaxial plain stress. Loads were applied to surface nodes located on the plate's surface as shown in Figure 7.

Results of simulation

As a result of the uniaxial plain stress induced in the plate by PZTs action, a single dimple is first created in the plate. Figure 8 shows this initial dimple resulting from Poisson's contraction.

With the increase in time, this initial dimple is transformed into changing pattern of plate surface deformations. This is probably due to rapidly propagating, as an elastic wave, of the initial dimple across the plate's surface. Figures 9 to 12 show the progress of transformation of the initial dimple into a pattern of deformations on the plate's surface.

It is easy to observe from the above images taken from ANSYS transient analysis that the deformation introduced to the plate by the initial dimple is rapidly spreading over the plate's surface.

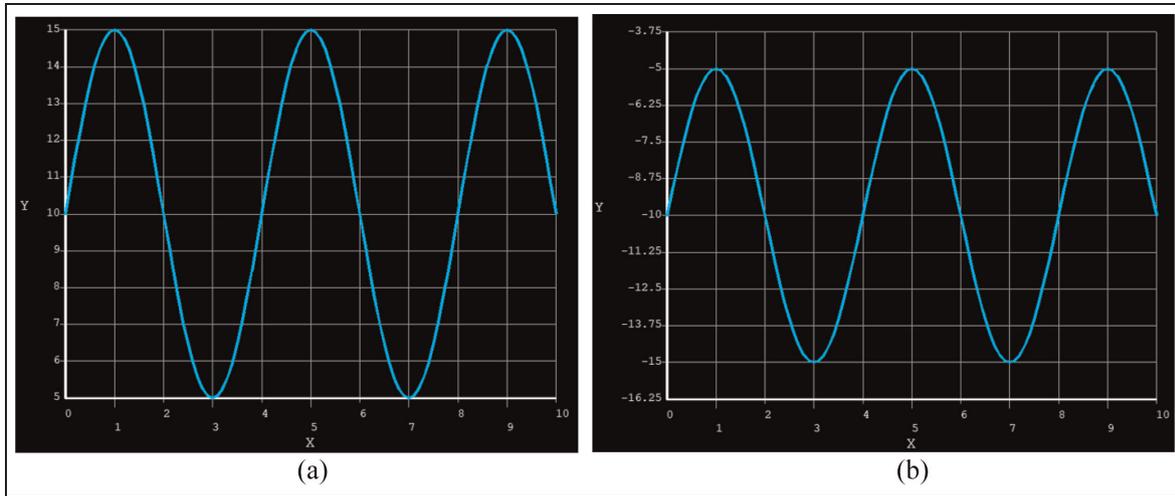


Figure 6. Sinusoidal fluctuation of the load generated by PZTs attached to the plate: (a) tensile loading in positive X and Y directions and (b) tensile loading in negative X and Y directions.

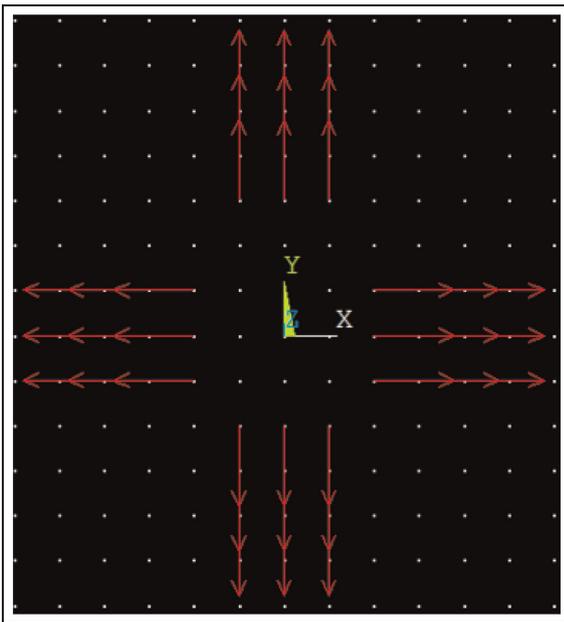


Figure 7. Computer simulation image showing arrangement of loads created by PZTs and attached to surface nodes.

Therefore, it is justified to state that the Poisson’s contraction is in a way a “prime mover” of the squeeze-film mechanism starting from a single, centrally located shallow dimple and ending with a pattern of elastic deformations of the plate.

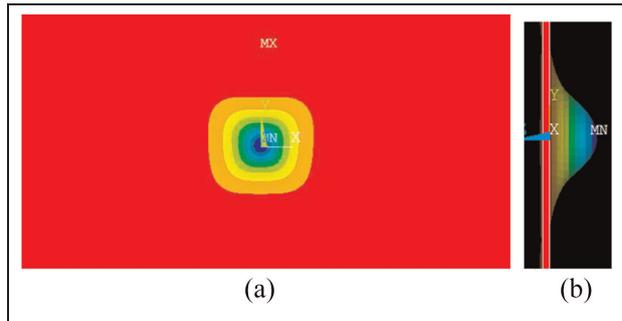


Figure 8. Initial dimple generated on the plate’s surface: (a) frontal view and (b) side view.

Concluding remarks

Findings presented in this paper support the hypothesis that the Poisson’s contraction plays a central role in the levitation of a light object and is a “prime mover” of the squeeze-film mechanism. Therefore, it can be asserted that the squeeze-film action induced by Poisson’s contraction can be utilised to levitate light objects not only in the vicinity of the initial central dimple but over entire surface of the plate. This can be taken as an explanation of experimentally observed fact that a light object floats not only at the centre of the plate where the initial dimple is created but also at other location on the plate’s surface.

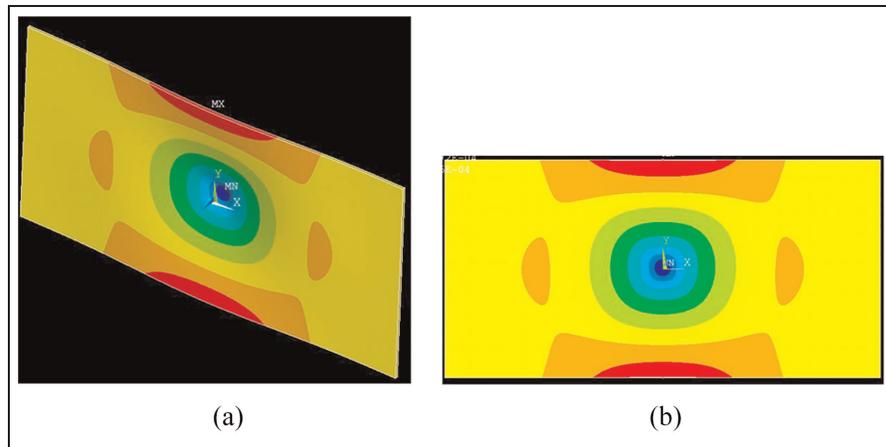


Figure 9. Transformation of the initial dimple after 0.0003 s: (a) isometric view and (b) frontal view.

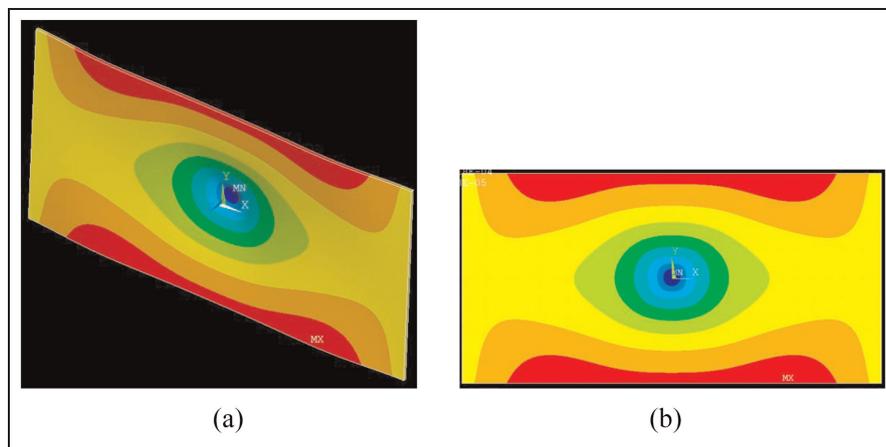


Figure 10. Transformation of the initial dimple after 0.0005 s: (a) isometric view and (b) frontal view.

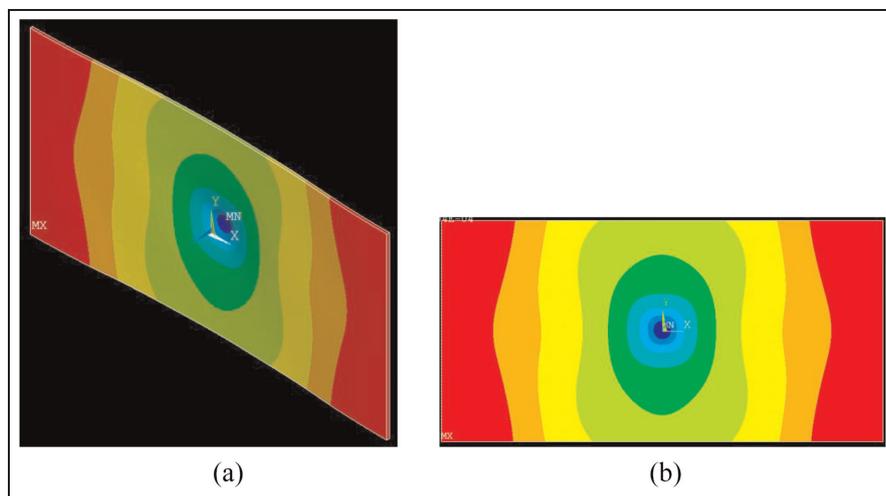


Figure 11. Transformation of the initial dimple after 0.0025 s: (a) isometric view and (b) frontal view.

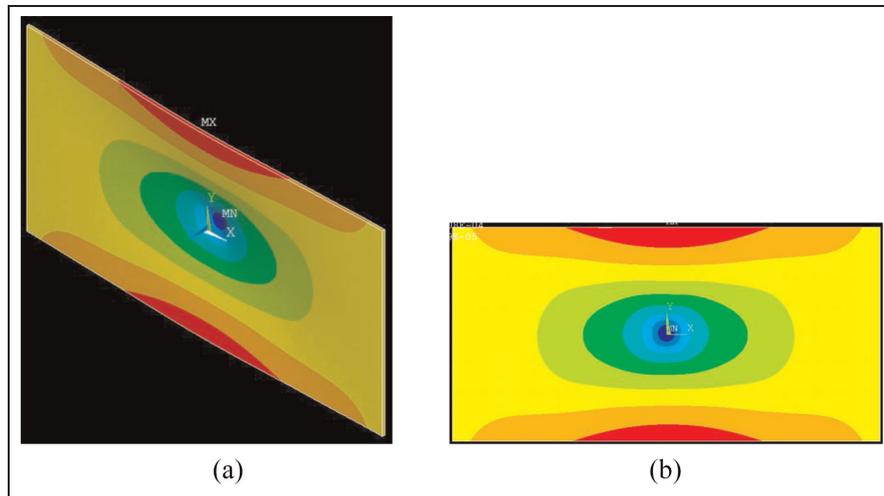


Figure 12. Transformation of the initial dimple after 0.0047 s: (a) isometric view and (b) frontal view.

Acknowledgement

The paper is a result of research collaboration between Department of Mechanical and Aerospace Engineering, Brunel University London and Department of Mechanical Engineering, Tokyo University of Science. Authors gratefully acknowledge the support given by both universities.

Declaration of conflicting interests

The author(s) declared no potential conflicts of interest with respect to the research, authorship, and/or publication of this article.

Funding

The author(s) received no financial support for the research, authorship, and/or publication of this article.

ORCID iD

Tadeusz Stolarski  <https://orcid.org/0000-0002-0090-0127>

References

1. Taniguchi N. *Nanotechnology: integrated processing systems for ultra-precision and ultra-fine products*. Oxford: Oxford University Press, 1996.
2. Fayed ME and Skocir T. *Mechanical conveyors: selection and operation*. Boca Raton, USA: Taylor and Francis, 1996.
3. McGuire PM. *Conveyors: application, selection, and integration*. 1st ed. Boca Raton, USA: Taylor and Francis, 2009.
4. Reinhart G and Hoepfner J. Non-contact handling using high-intensity ultrasonics. *CIRP Ann* 2000; 49: 5–8.
5. Vandaele V, Lambert P and Delchambre A. Non-contact handling in micro assembly: acoustical levitation. *Precis Eng* 2005; 29: 491–505.
6. Liu P, Li J, Ding H, et al. Modelling and experimental study on near-field acoustic levitation by flexural mode. *IEEE Trans Ultrason Ferroelectr Freq Control* 2009; 56: 87–93.
7. Reinhart G and Hoepfner J. Non-contact handling using high-intensity ultrasonics. *CIRP Ann* 2000; 49: 5–8.
8. Stolarski TA and Chai W. Self-levitating sliding air contact. *Int J Mech Sci* 2006; 48: 601–620.
9. Shou T, Yoshimoto S and Stolarski T. Running performance of an aerodynamic journal bearing with squeeze film effect. *Int J Mech Sci* 2013; 77: 184–193.
10. Langlois WE. Isothermal squeeze films. *Q Appl Math* 1962; 20: 131–150.
11. Salbu E. Compressible squeeze films and squeeze bearings. *J Basic Eng* 1964; 86: 355–364.
12. Whymark R. Acoustic field positioning for container-less processing. *Ultrasonics* 1975; 13: 251–261.
13. Wiesendanger M. *Squeeze film air bearings using piezoelectric bending elements*. Thesis, Ecole Polytechnique Federal de Lausanne, Switzerland, 2001.
14. Stolarski TA. Numerical modelling and experimental verification of compressible squeeze film pressure. *Tribol Int* 2010; 43: 356–360.
15. Atherton MA, Mares C and Stolarski TA. Some fundamental aspects of self-levitating sliding contact bearings and their practical implementations. *Proc IMechE, Part J: J Engineering Tribology* 2014; 22: 916–927.
16. Stolarski T and Chai W. Inertia effect in squeeze film air contact. *Tribol Int* 2008; 41: 716–723.
17. *Poisson's ratio*. Wikipedia, https://en.wikipedia.org/wiki/Poisson's_ratio.
18. Woolliscroft CI. *Ultrasonic acoustic levitation*. BEng Dissertation, Mechanical Engineering, School of Engineering and Design, Brunel University, 2005.